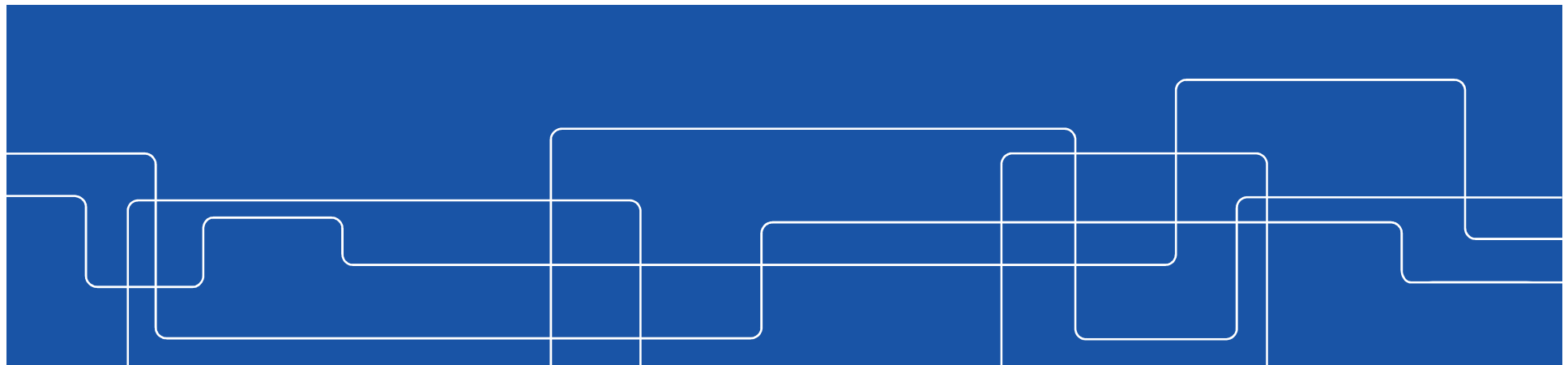




Dynamic Exhaust Valve Flow 1-D Modelling during Blowdown Conditions

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11.10.2018, CCGEx – Research Day



VOLVO



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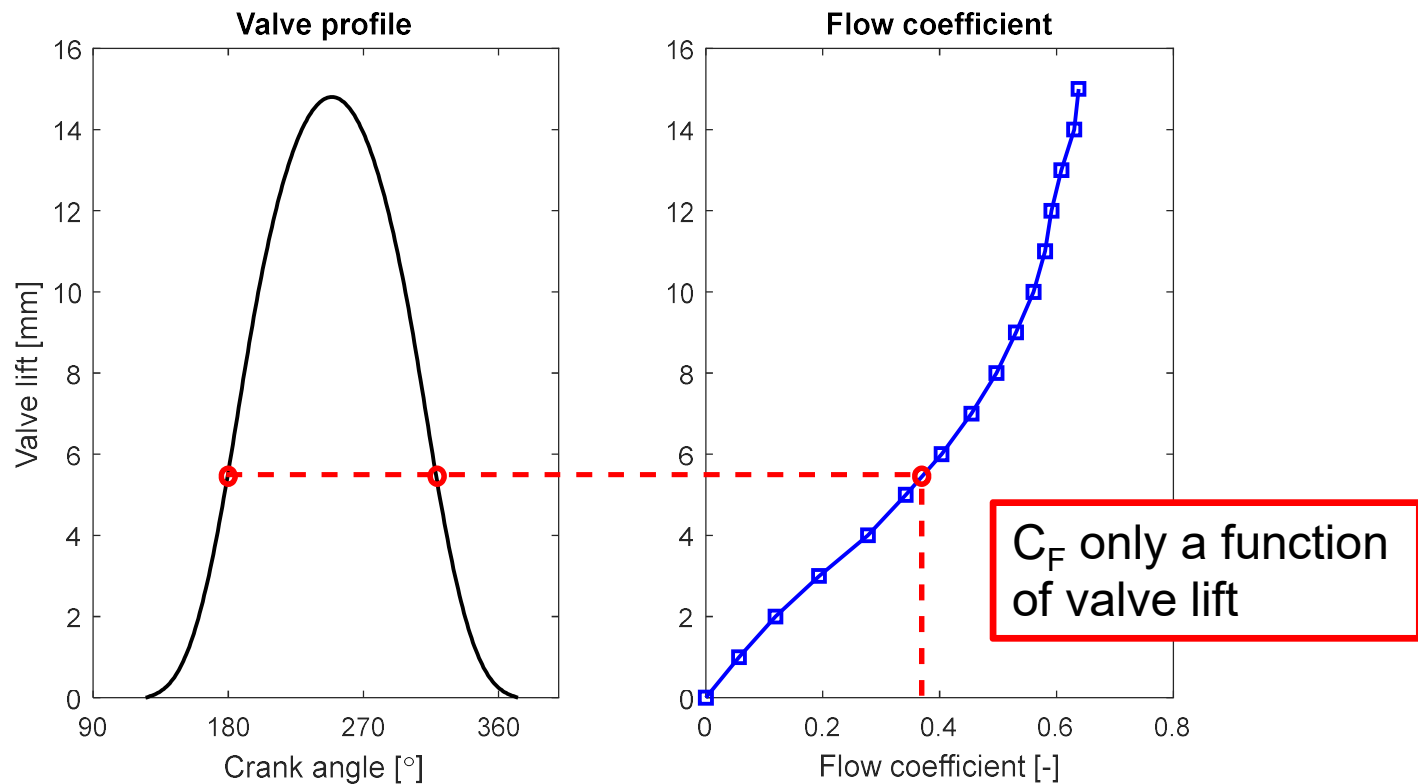


Project

- Valve strategies and exhaust pulse utilization
- 1-D GT-Power
- Engine experiments

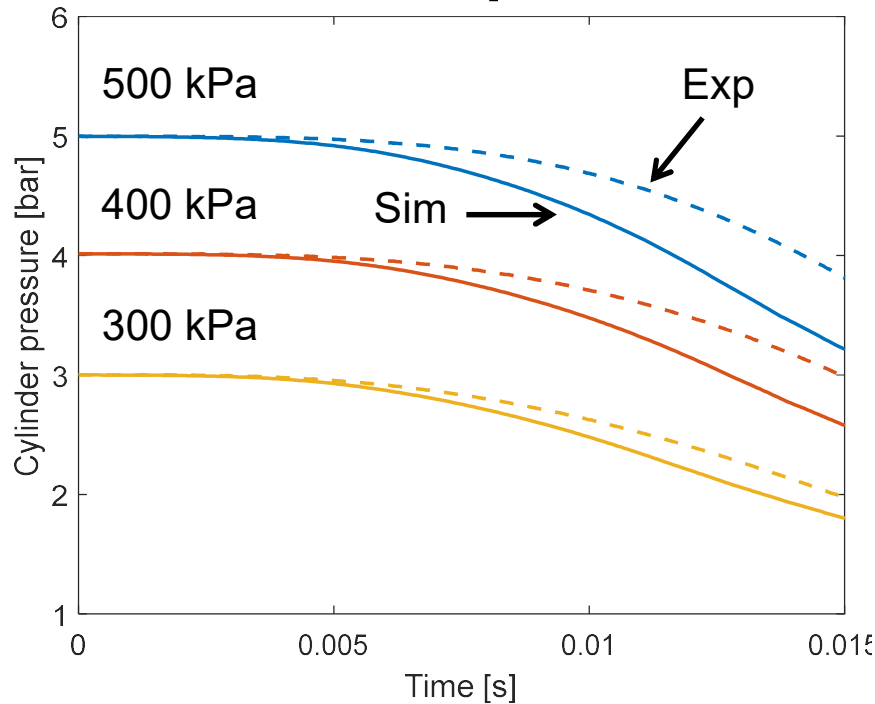
Valve flow coefficient

- 1D flow: quasi-steady assumption

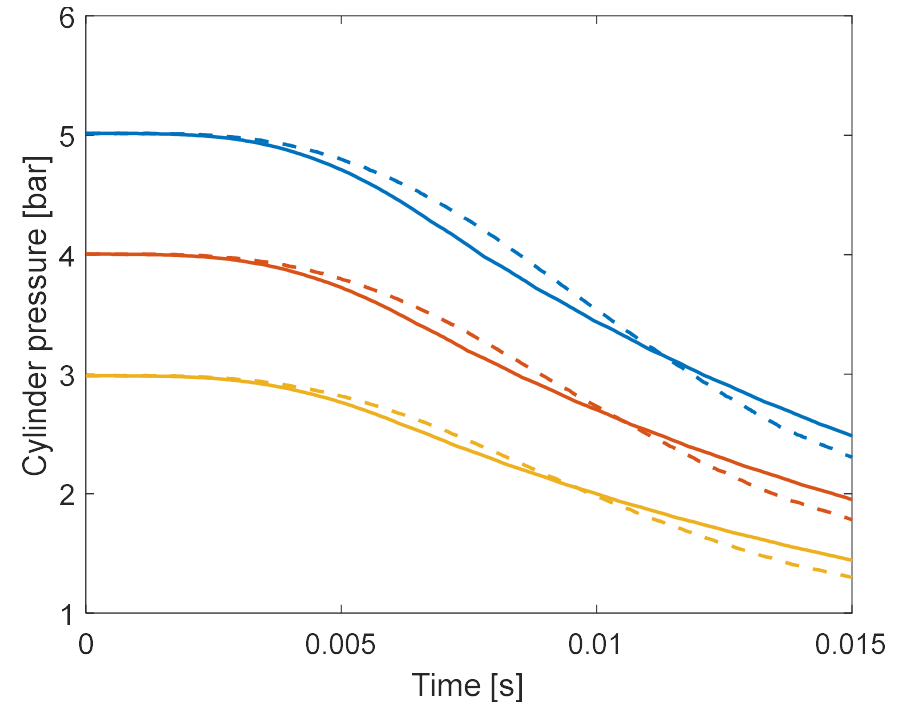


Simulation vs Experiments

800 rpm



1350 rpm

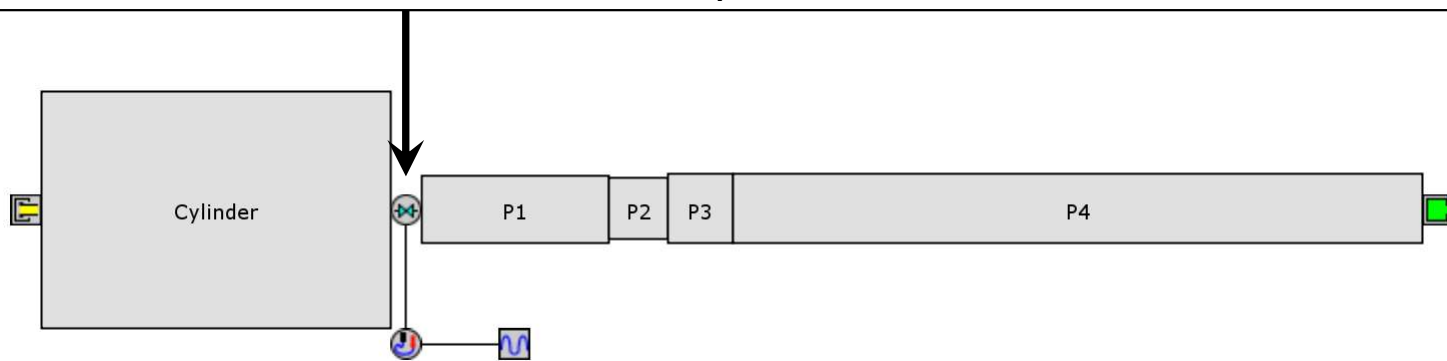


- ❑ QS valve flow over predicts the rate of cylinder emptying
- ❑ C_F function of valve opening speed and initial cylinder pressure

GT-Power model

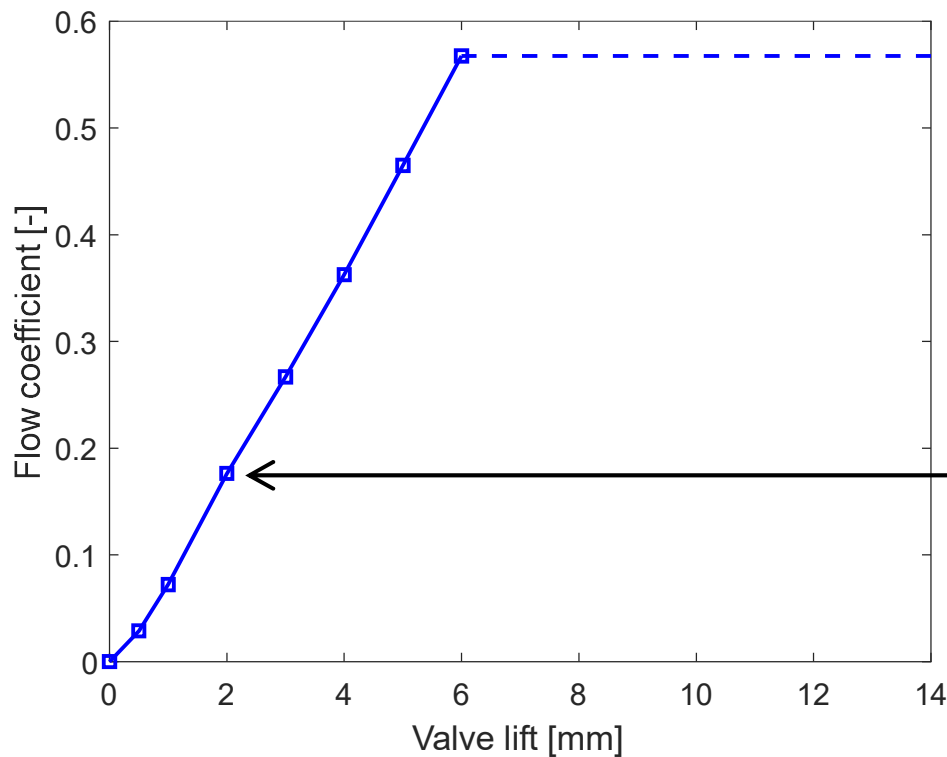
- ❑ Replicate the experimental conditions
 - ❑ Initial cylinder pressure
 - ❑ Volumes and pipe dimensions

Valve object: Steady-flow C_F from the same setup

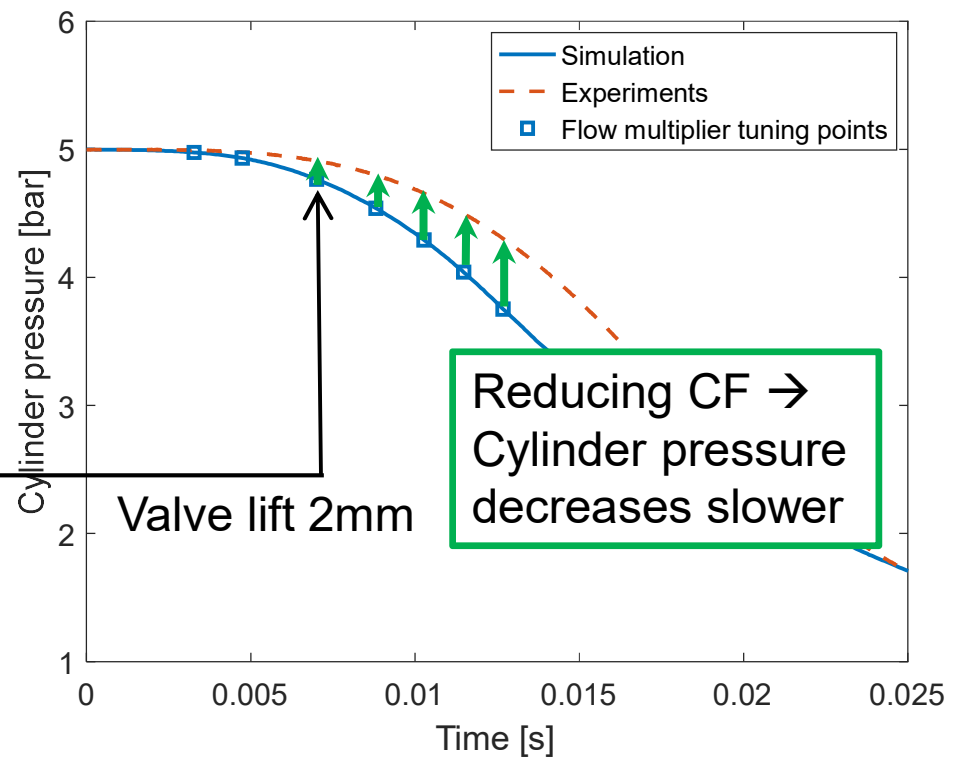


Measured valve profile imposed

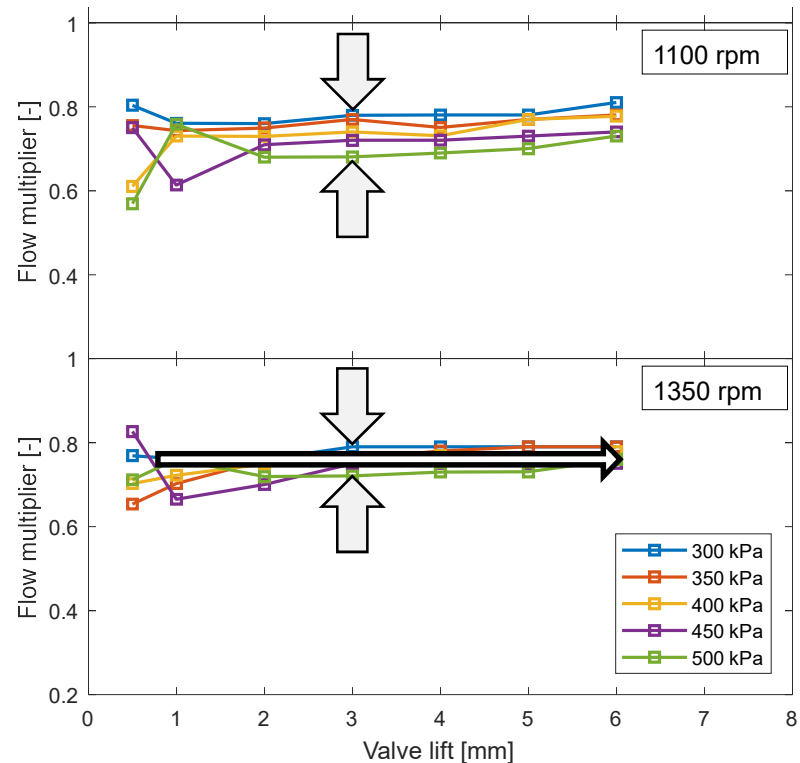
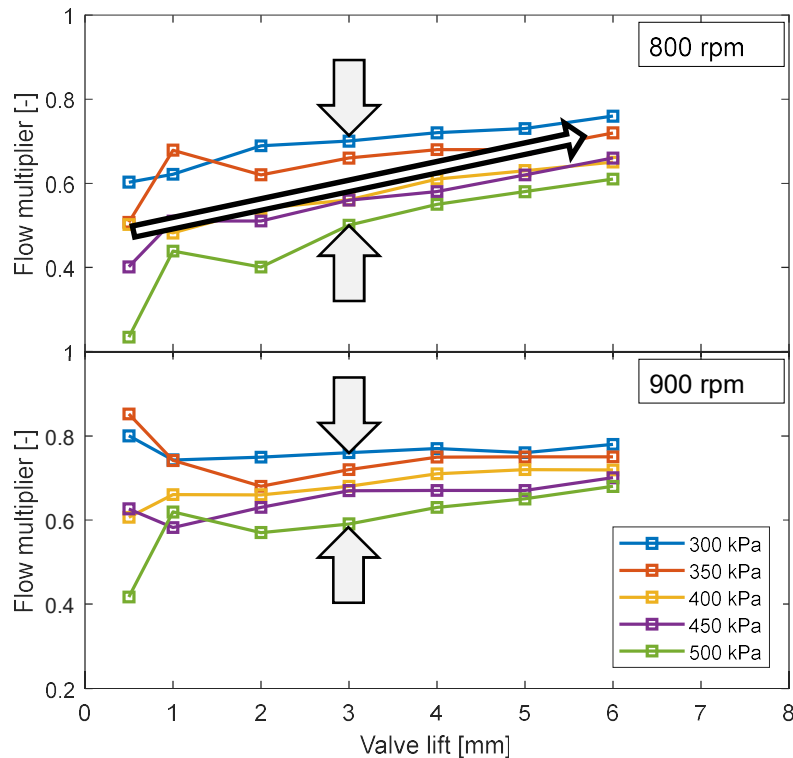
Steady-flow valve flow coefficients
(Choked flow conditions)



Cylinder emptying



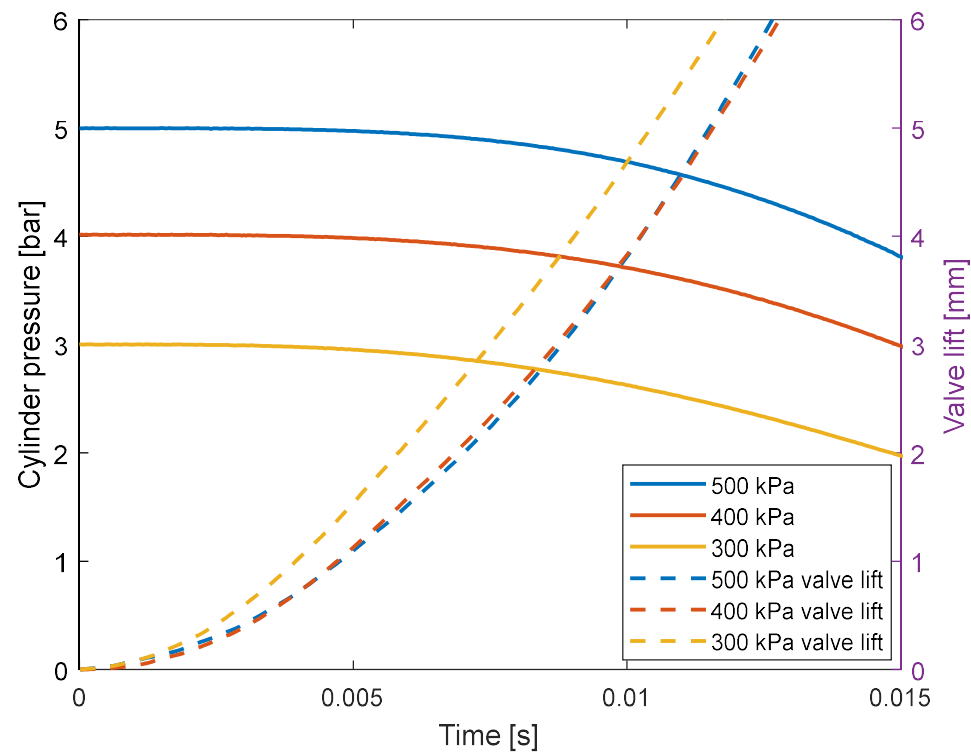
Flow multiplier (fraction of steady-flow C_F)



- ❑ Influence of initial pressure reduces with valve speed
- ❑ Influence of valve lift reduces with valve speed

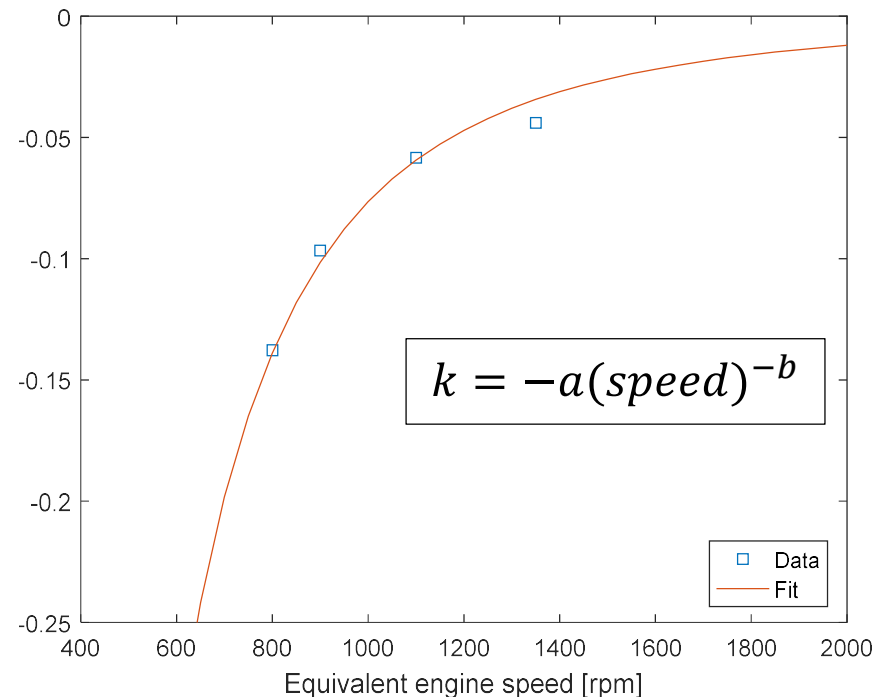
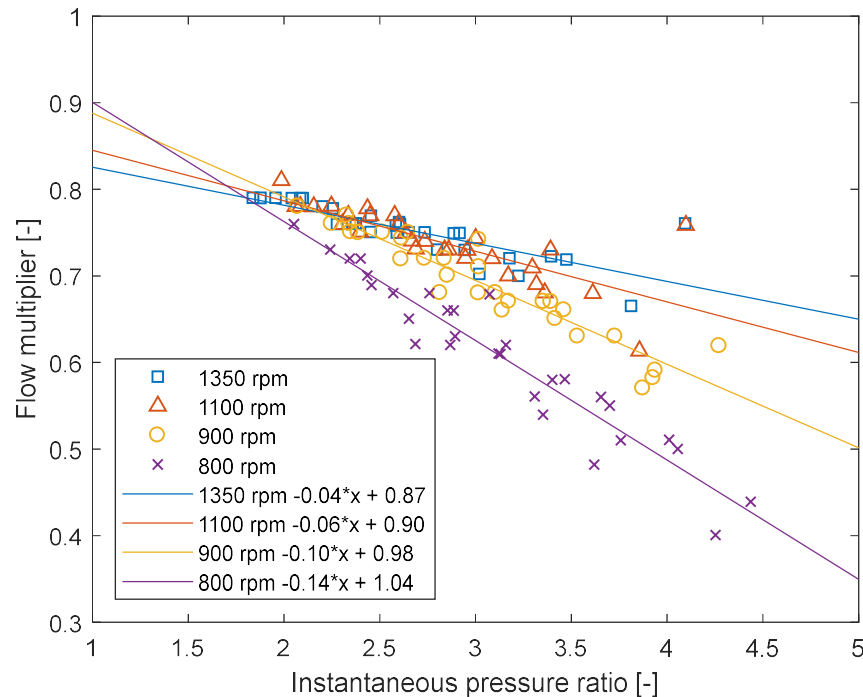
Flow multiplier

- At the same valve lift, the instantaneous pressure ratio is different for each test case



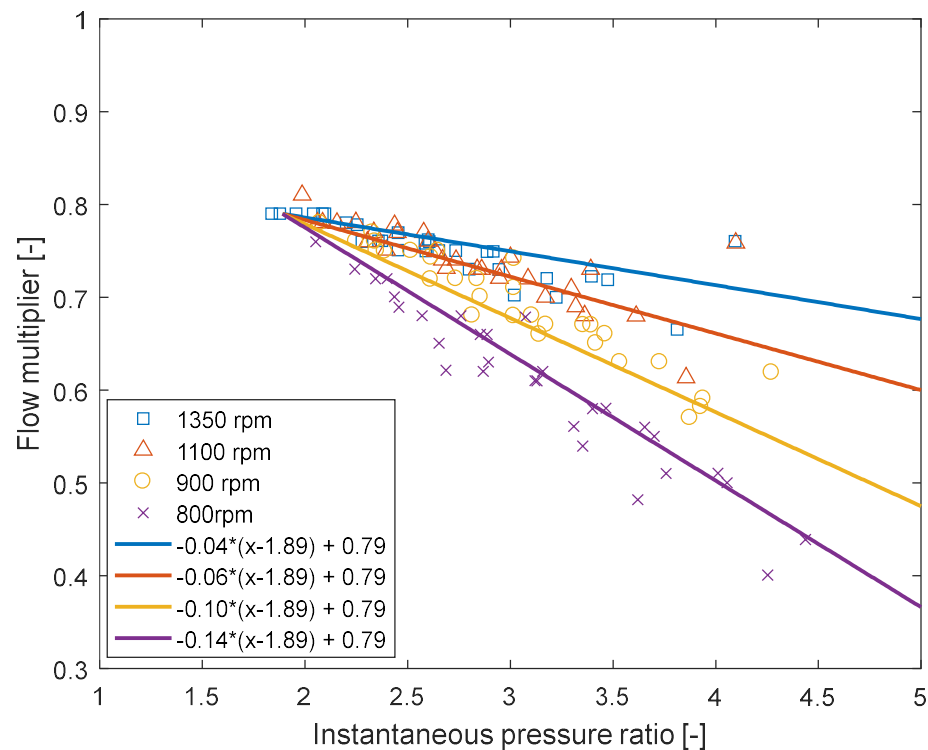
Flow multiplier model

- GTP → Flow multiplier & instantaneous PR at a given lift
- Plot all initial pressures together



Final model

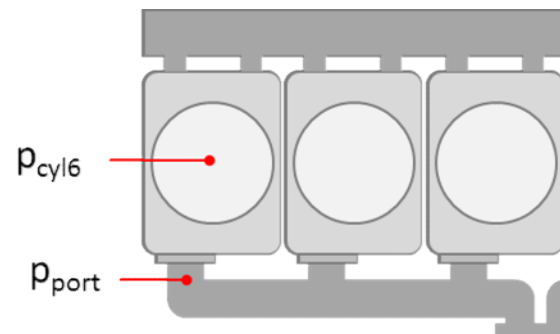
- Dependent on engine speed and pressure ratio
- Origin point $(x_0, y_0) = (1.89, 0.79)$



$$k = -a(\text{speed})^{-b}$$

$$y = k(PR - x_0) + y_0$$

- ❑ Implementing flow multiplier model in a GT-Power engine model to investigate impact on performance
- ❑ Engine test to measure valve flow through fast cylinder pressure and exhaust port measurement





Competence Center for Gas Exchange



”Charging for the future”



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