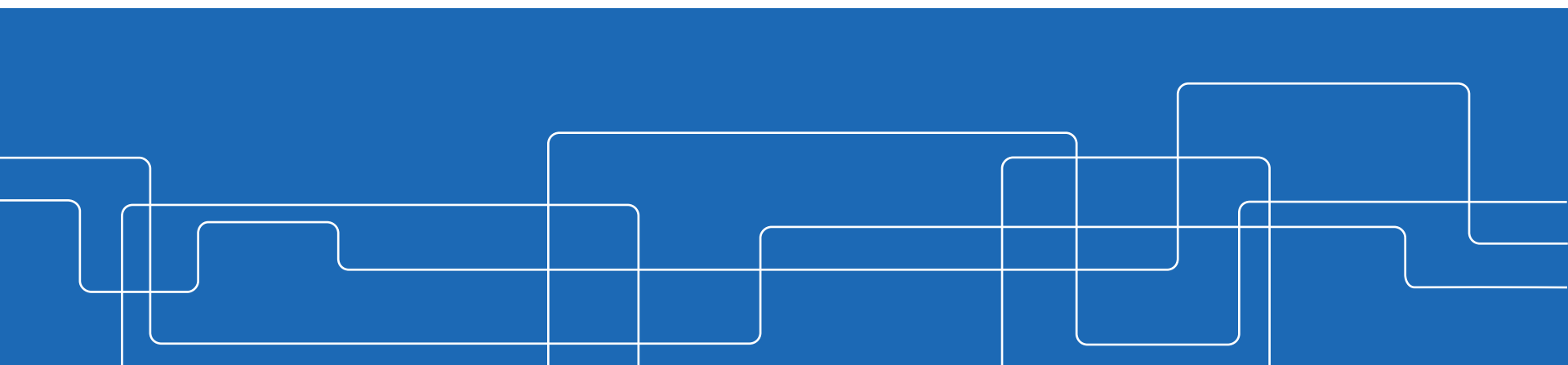


Gas dynamics of exhaust valves

Marcus Winroth, Henrik Alfredsson
11.10.2018, CCGEx - Research days



VOLVO



BorgWarner

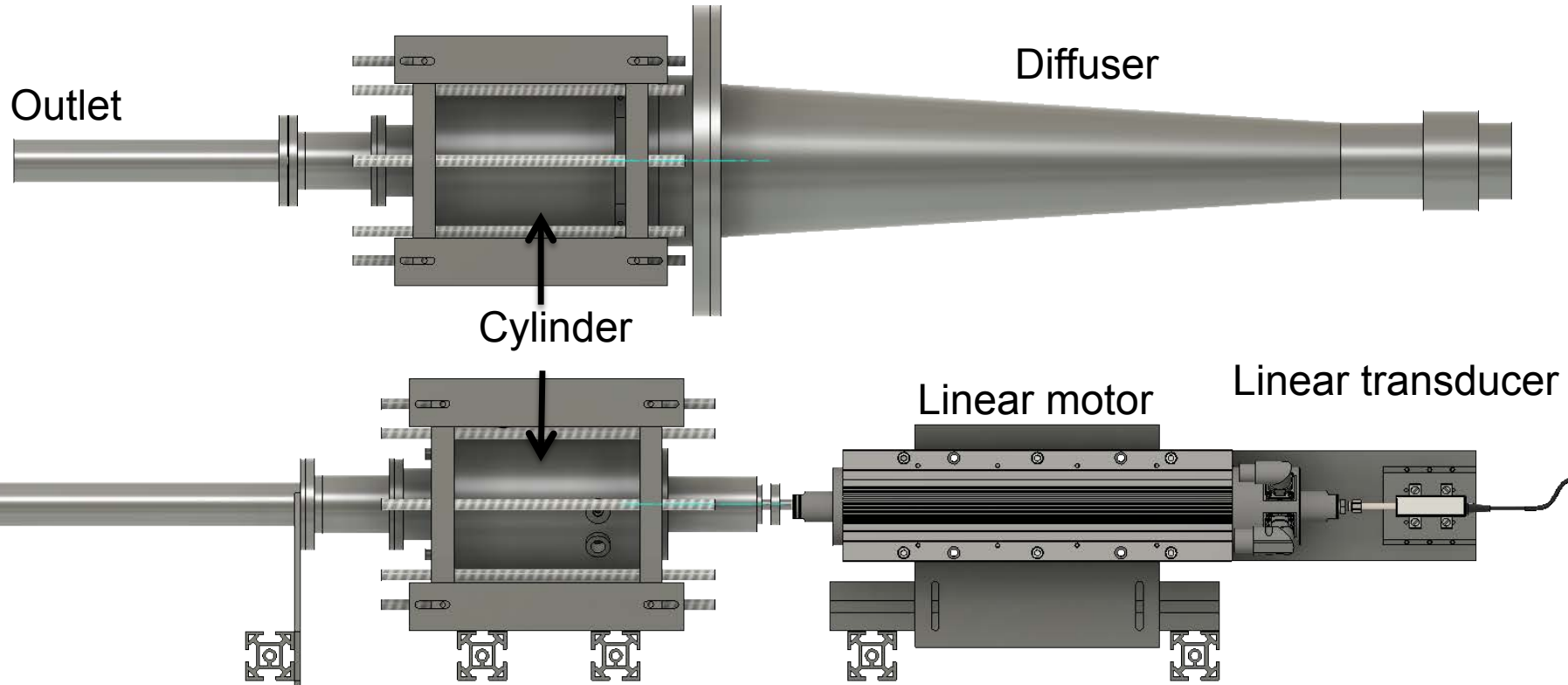


Outline



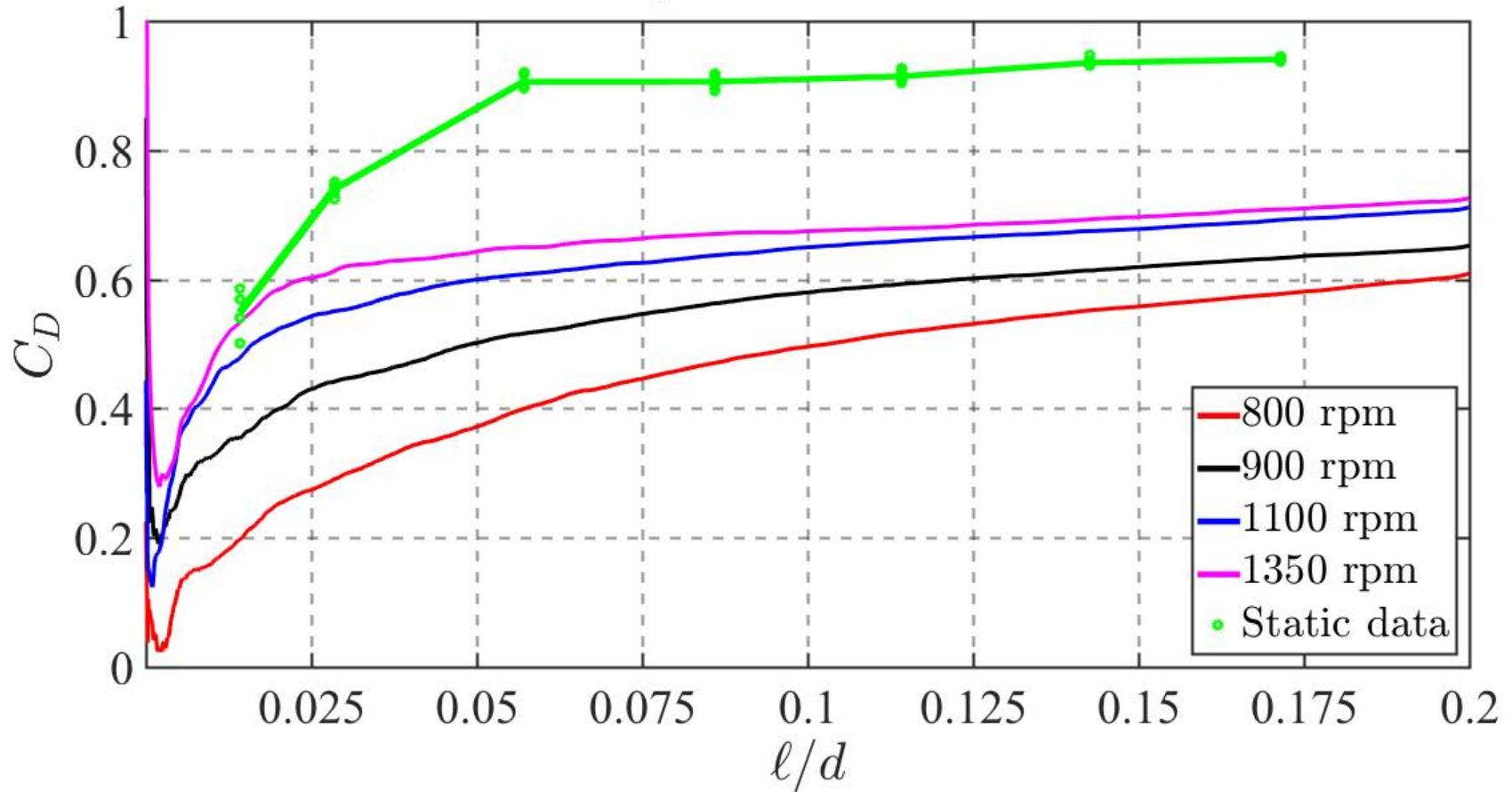
- ❑ Previous experiments – steady state vs dynamic valve C_D measurements
- ❑ Gas dynamics during valve opening
 - New flow visualization setup
 - Principle of Schlieren visualization
 - Flow physics in the exhaust port

Previous experiments



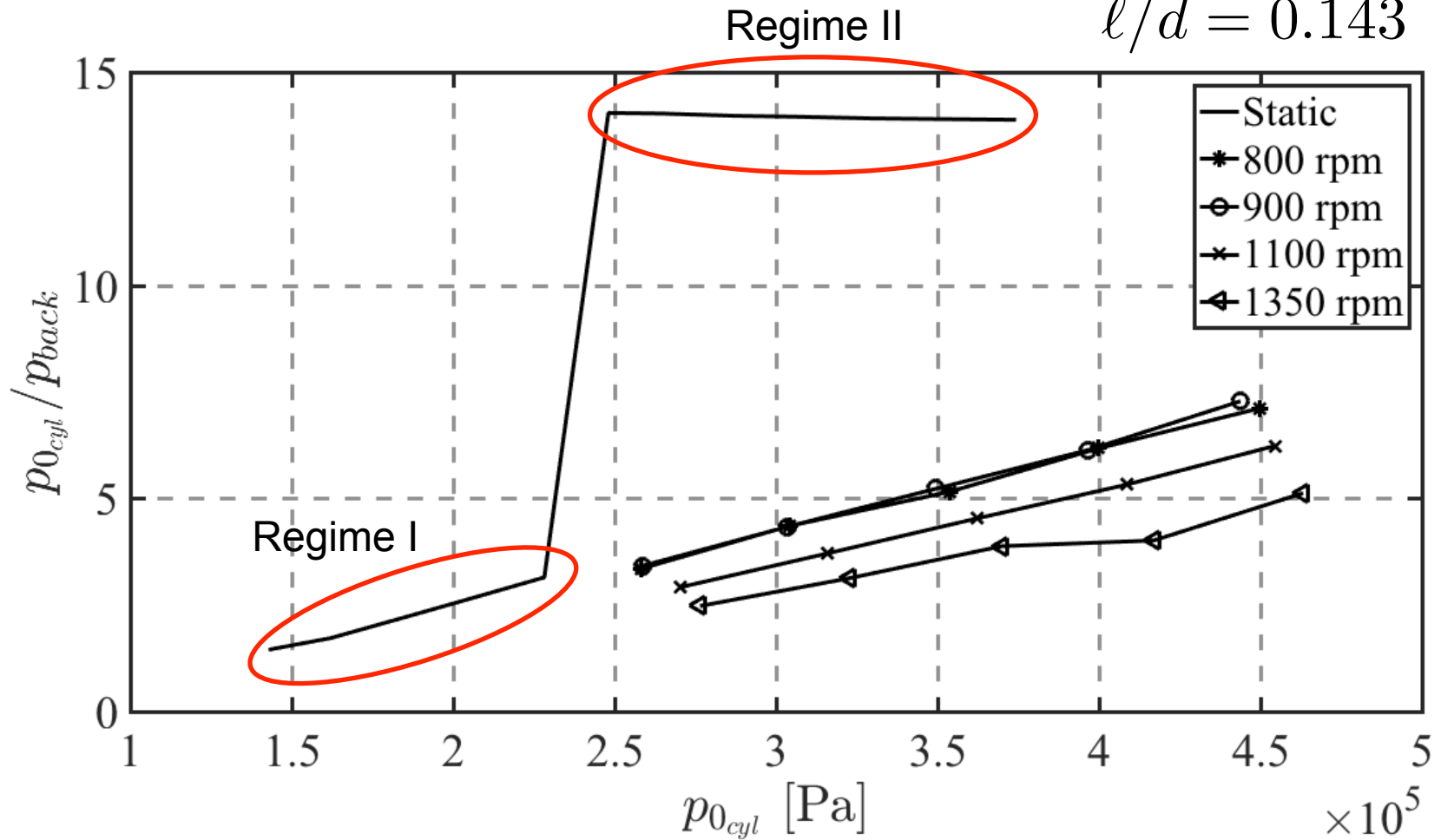
C_D for static and dynamic cases

$p_{0i} = 500 \text{ kPa}$

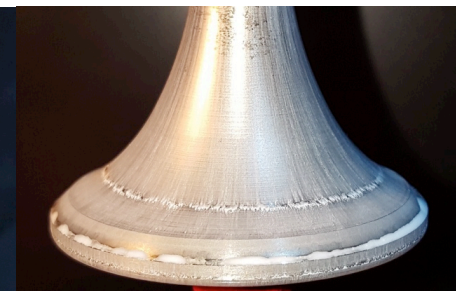
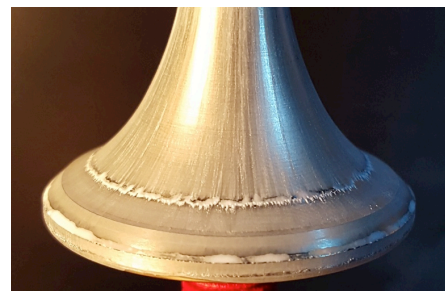
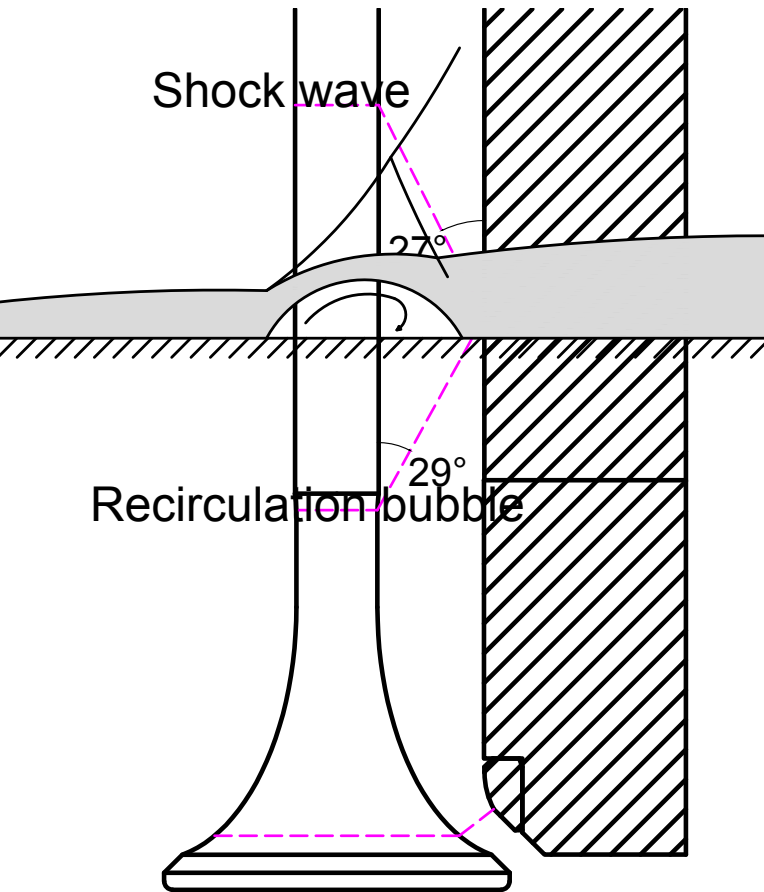


Cylinder to back pressure ratio

$$\ell/d = 0.143$$



Surface flow visualisation



Regime I

Regime II

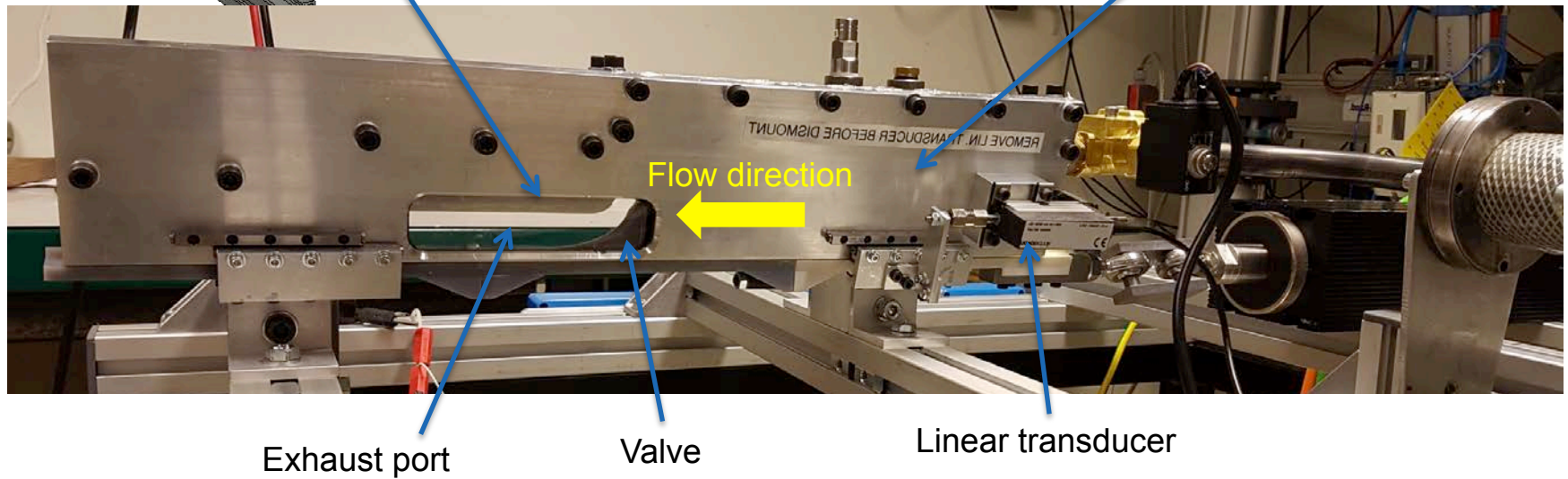
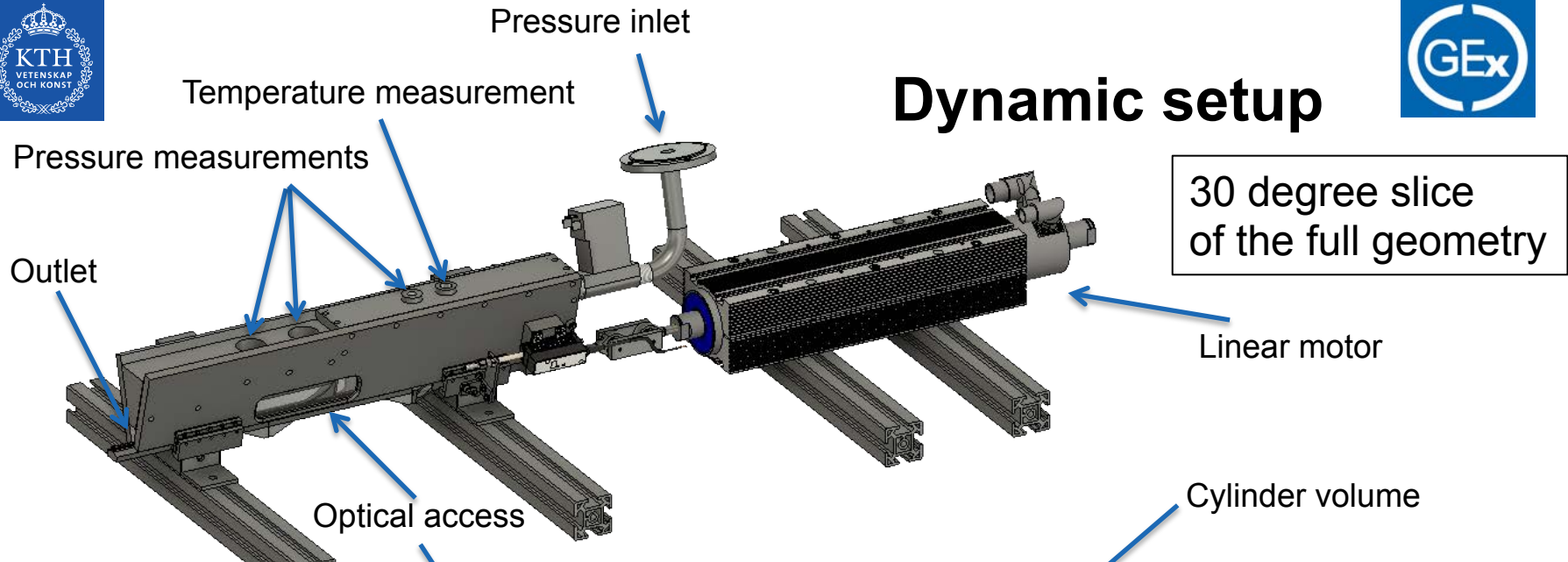


Outline

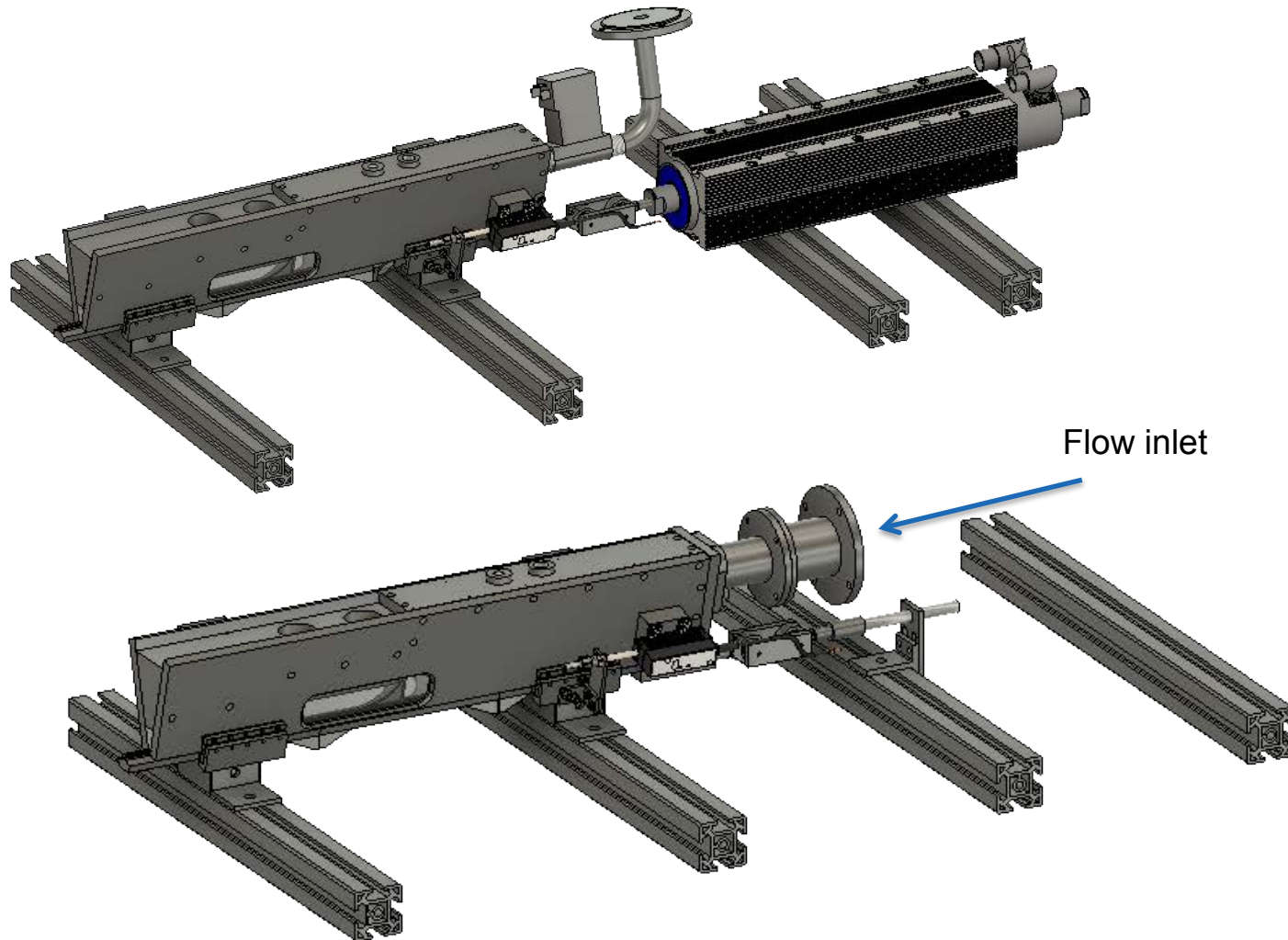


- Previous experiments – steady state vs dynamic valve C_D measurements
- Gas dynamics during valve opening
 - New flow visualization setup
 - Principle of Schlieren visualization
 - Flow physics in the exhaust port

Dynamic setup



Static setup



Schlieren setup

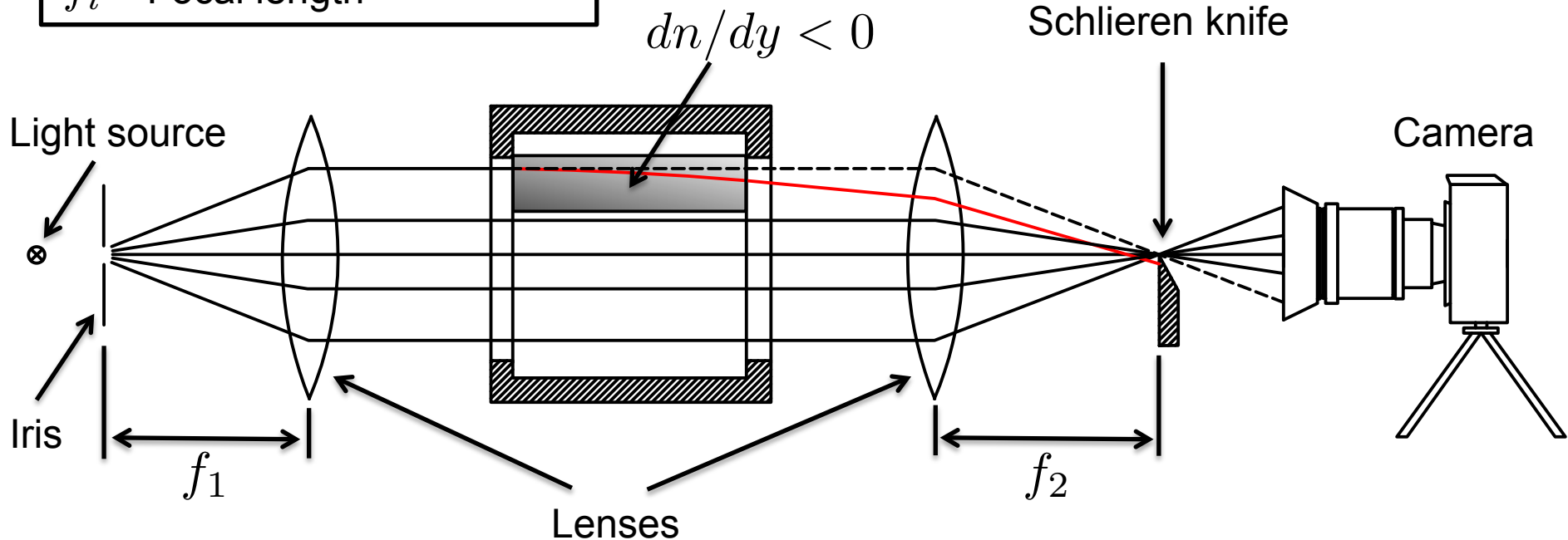
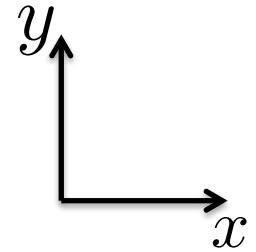
n - Refractive index

ρ - Density

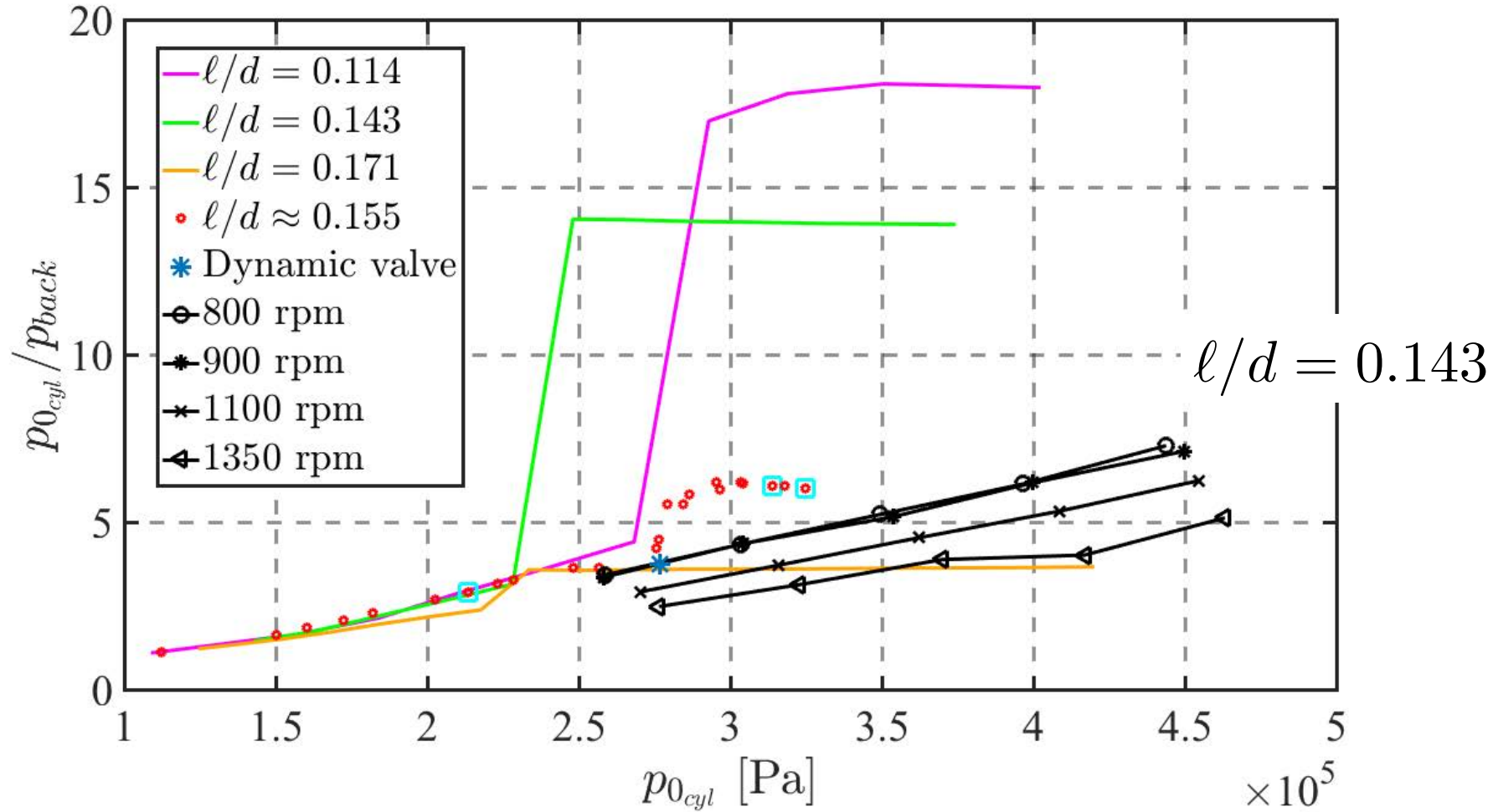
K - Gladstone-Dale constant

f_i - Focal length

$$n = 1 + K\rho$$

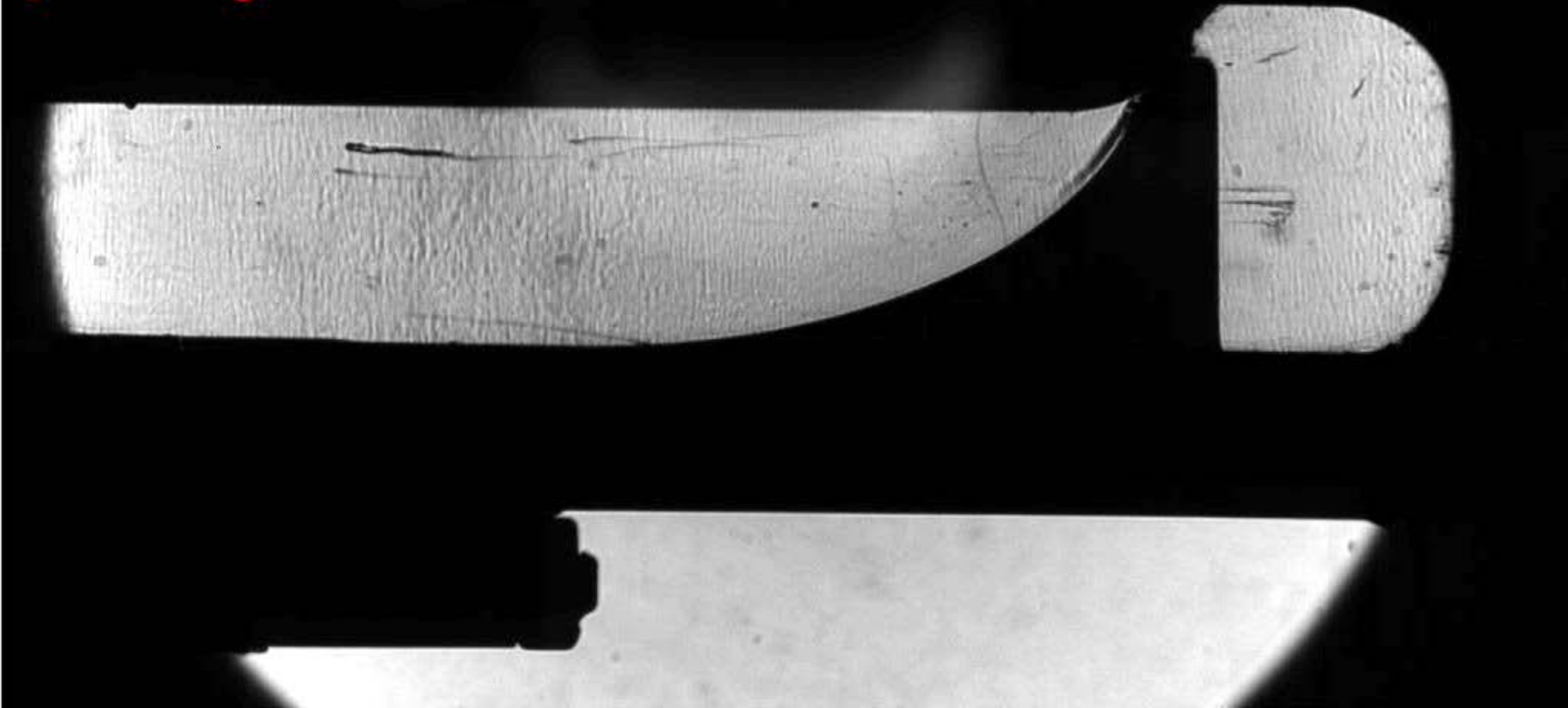


Pressure ratio



Dynamic process ($n = 1350$ rpm)

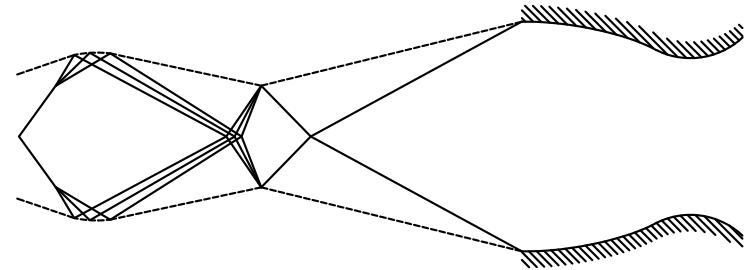
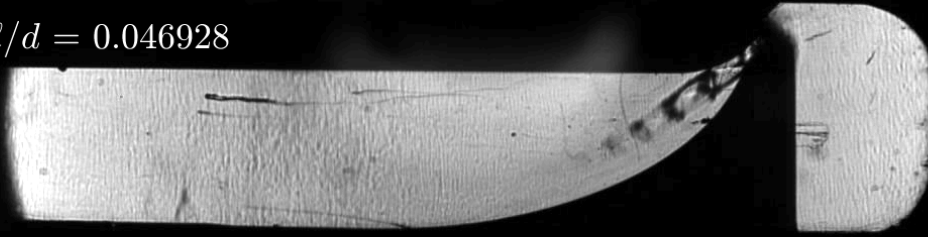
$t = 0$



Flow states in the dynamic process

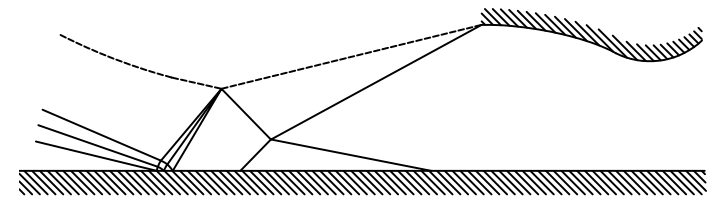
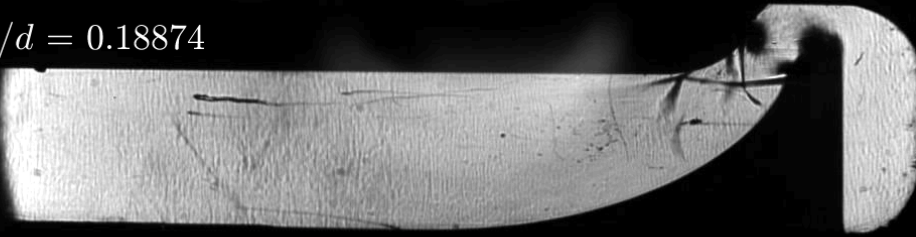
Dynamic valve

$$\ell/d = 0.046928$$



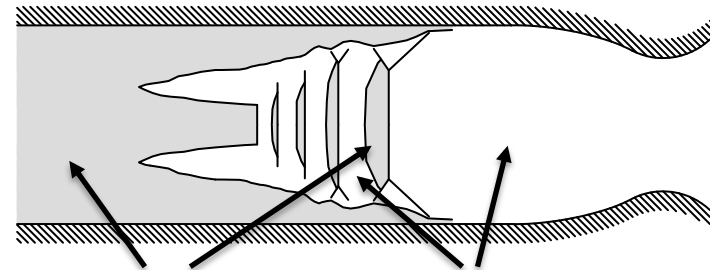
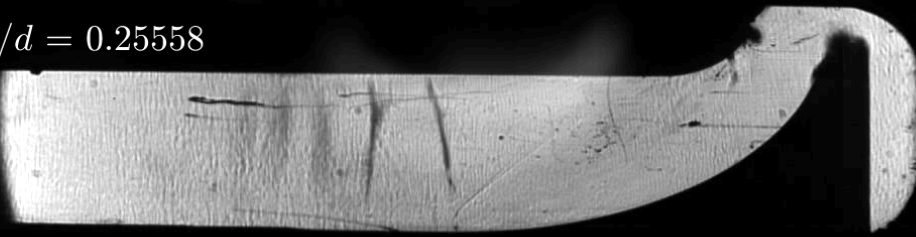
Dynamic valve

$$\ell/d = 0.18874$$



Dynamic valve

$$\ell/d = 0.25558$$



Subsonic

Supersonic

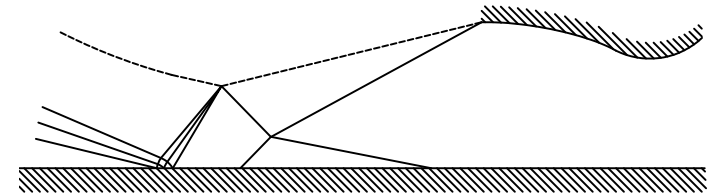
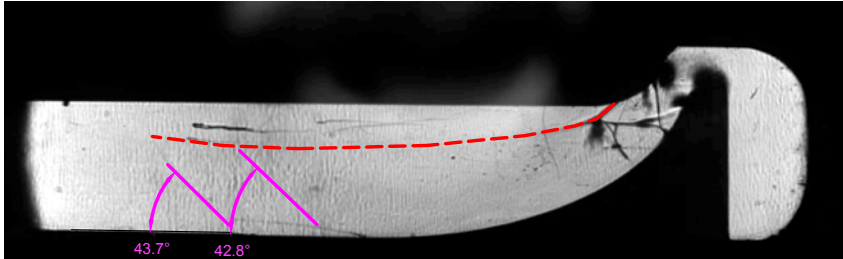
Comparing dynamic & static operations

Dynamic

$$\ell/d = 0.154$$

$$\mu = 42.8^\circ$$

$$M = 1.47$$



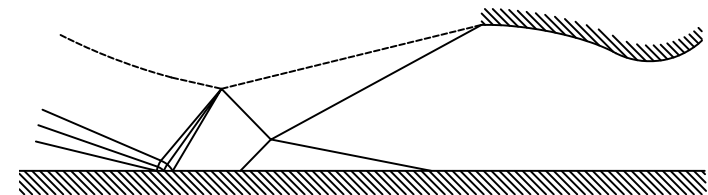
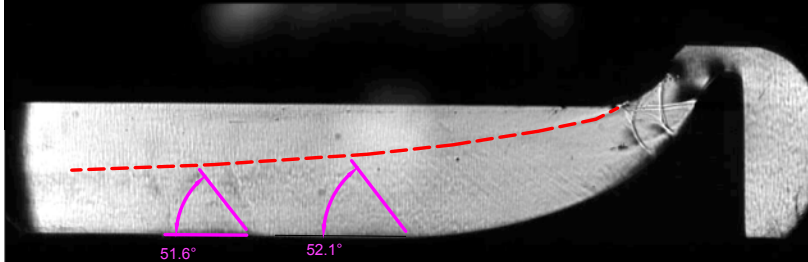
Static

Regime I

$$\ell/d = 0.155$$

$$\mu = 52.1^\circ$$

$$M = 1.27$$

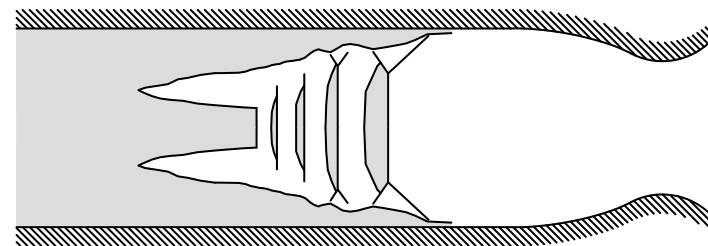
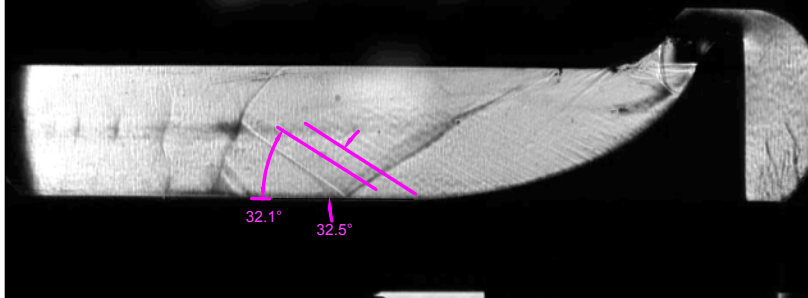


Regime II

$$\ell/d = 0.155$$

$$\mu = 32.1^\circ$$

$$M = 1.88$$



$$M = \frac{1}{\sin \mu}$$

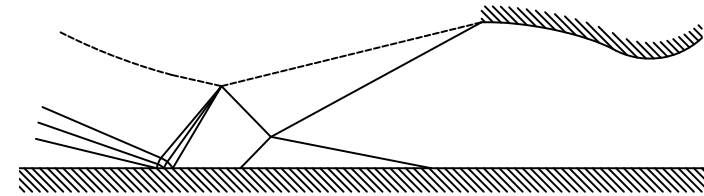
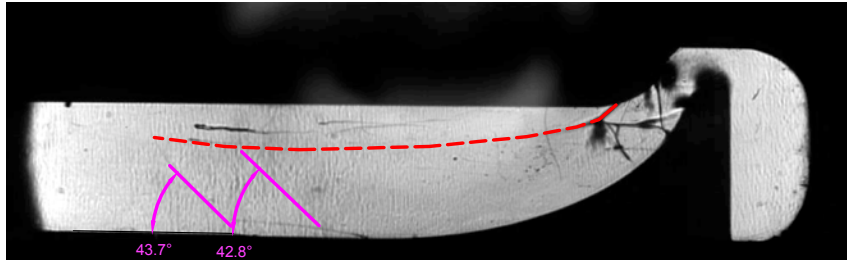
Comparing dynamic & static operations

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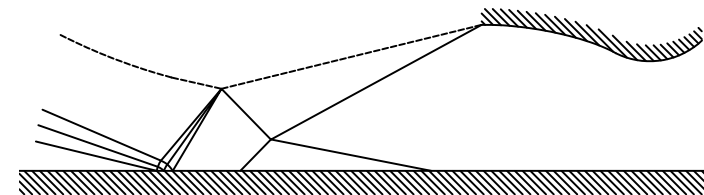
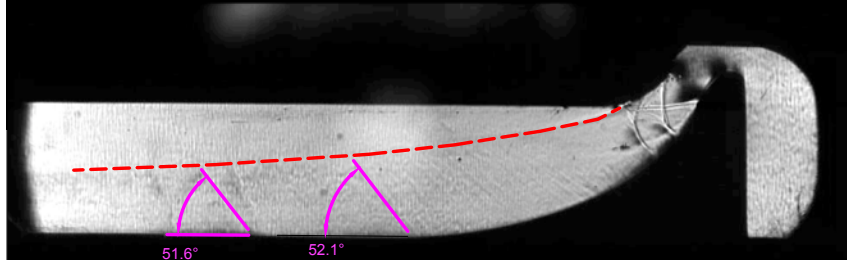
Static

Regime I

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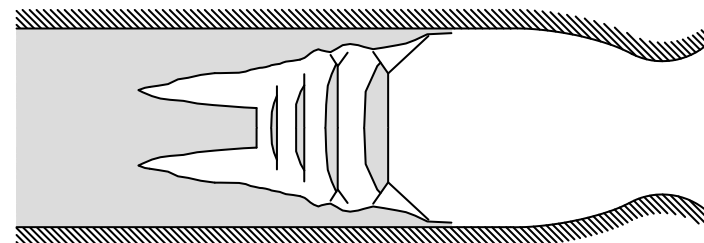
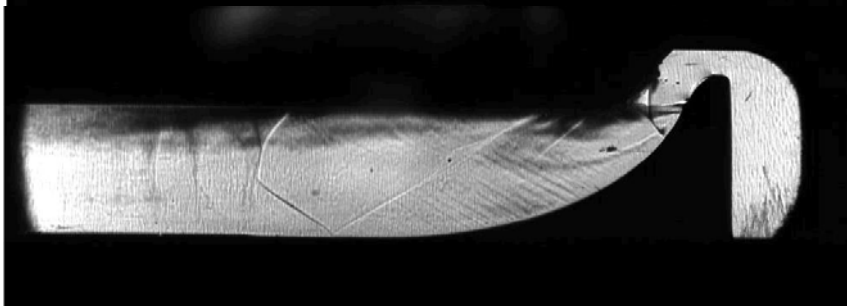


Regime II

$$\ell/d = 0.155$$

$$\mu = 32.1^\circ$$

$$M = 1.88$$



$$M = \frac{1}{\sin \mu}$$

Conclusions

- ❑ The exhaust port flow has three distinct flow states:
 - A. Overexpanded jet with free boundaries (small l/d)
 - B. Overexpanded wall bounded jet with one free boundary (medium l/d)
 - C. Fully expanded flow terminating in a normal pseudo-shock (large l/d)
- ❑ The dynamic discharge process goes through flow states A - C, showing mainly a dependency on l/d .
- ❑ The steady flow process show a transition from flow state B. (regime I) to flow state C. (regime II) (function of cylinder pressure) at lower l/d compared to the dynamic process.
- ❑ The characteristics of the state B jet (shock pattern, shape, Mach number) differs between dynamic and static operations.



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