



Aerothermodynamics and exergy analysis in turbocharger radial turbine

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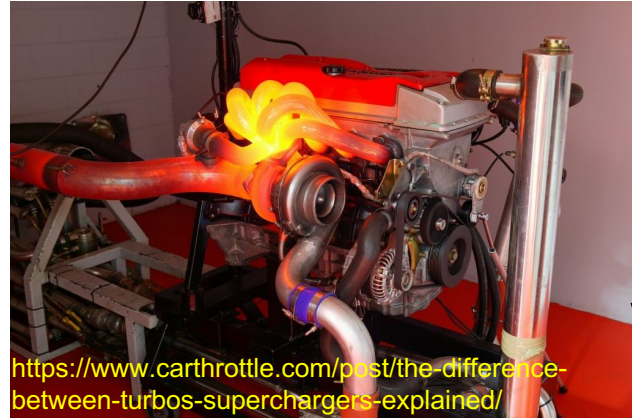
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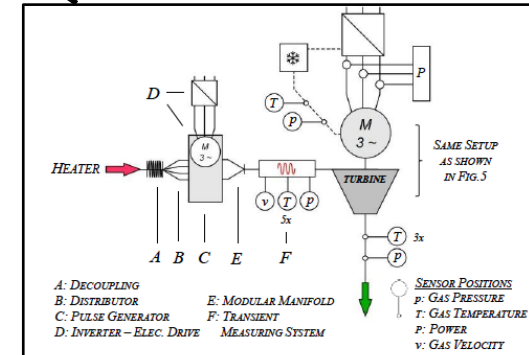
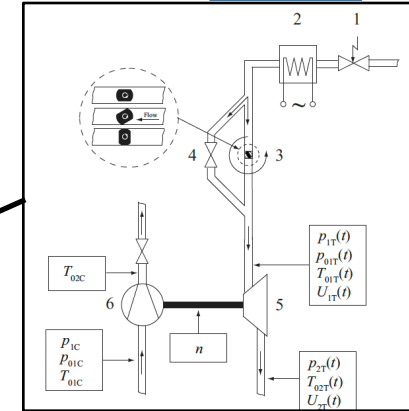
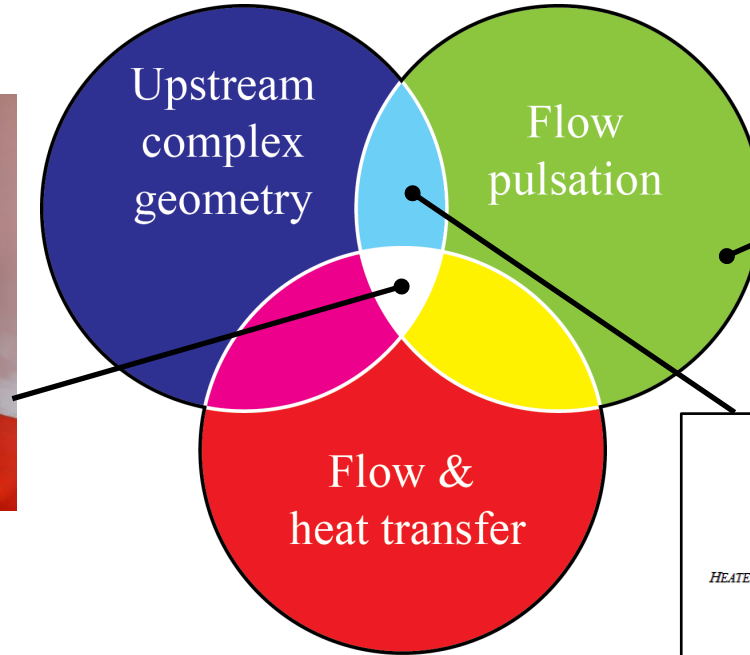
VOLVO



The turbocharger turbine



<https://www.carthrottle.com/post/the-difference-between-turbo-superchargers-explained/>

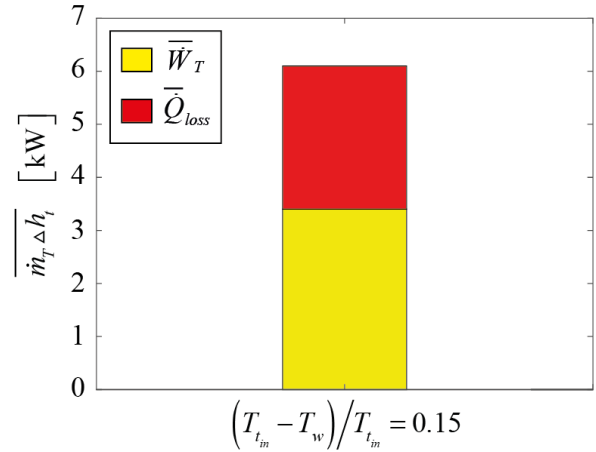
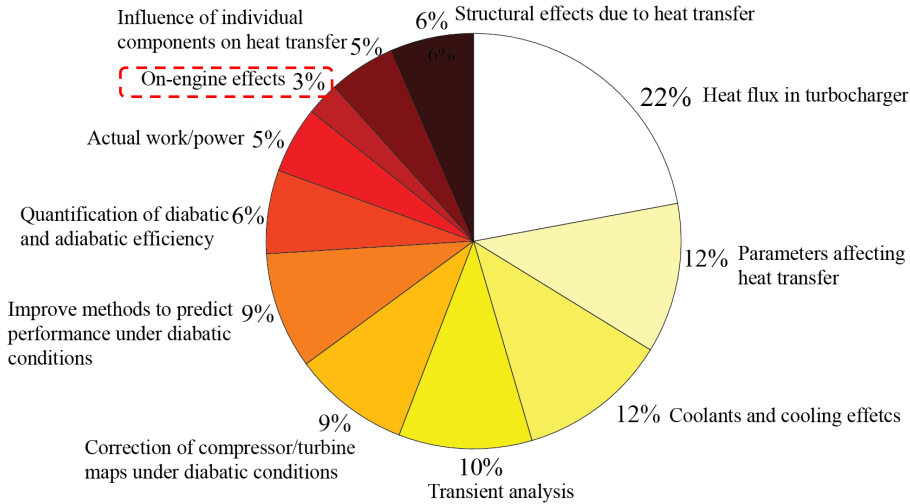


Laurantz F, Örlü R, Segalini A, Tillmark N, Alfredsson H. Experimental analysis of turbocharger interaction with a pulsatile flow through time-resolved flow measurements upstream and downstream of the turbine. In: Institution of Mechanical Engineers - 10th International Conference on Turbochargers and Turbocharging [Internet]. 2012. p. 405–15.

Lyttek P, Roclawski H, Böhle M, Gugau M. New Modular Test Rig for Unsteady Performance Assessment of Automotive Turbocharger Turbines. ASME. Turbo Expo: Power for Land, Sea, and Air, Volume 8: Microturbines, Turbochargers and Small Turbomachines ();V008T26A017. doi:10.1115/GT2017-64218.

1st law of thermodynamics

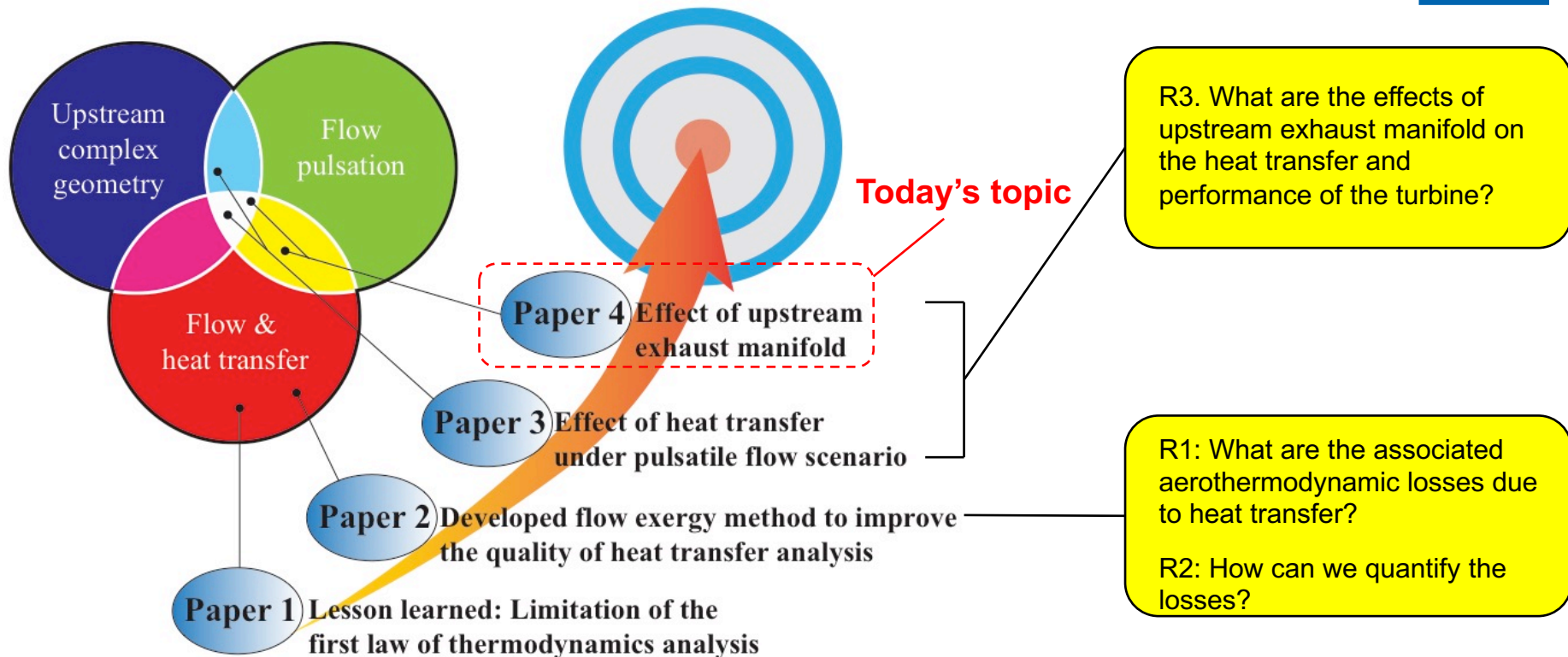
$$\Delta h = q - w$$



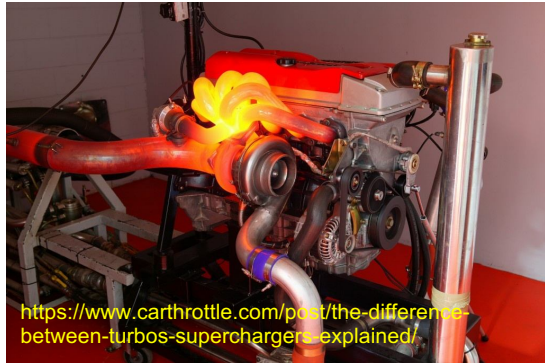
Aerothermodynamic losses?

- ❑ Limited insights into the thermo-fluid effects due to heat transfer with the first law of thermodynamic based method
- ❑ Customary set-up ignore the presence of the upstream exhaust manifold
- ❑ Limited knowledge about the thermo-fluid physics under on-engine conditions

Goal: Understand the thermo-fluid physics and the turbine performance under realistic engine conditions



Challenges

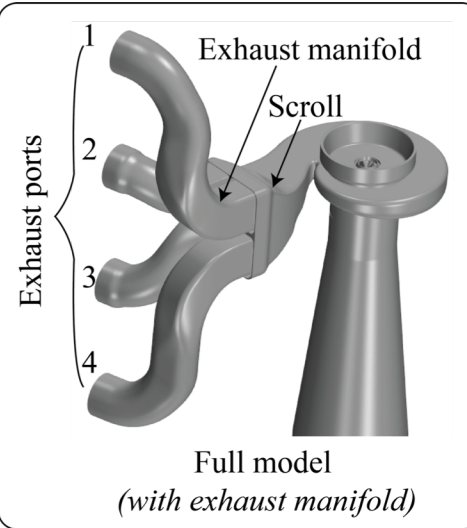


experiment
measurement/
visualization

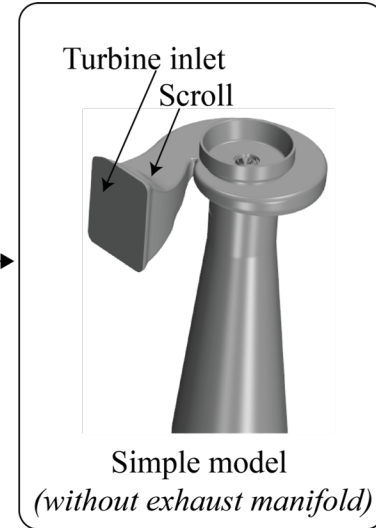
Strategy

1D model
(experimentally calibrated engine simulation model by industry partner)

BCs



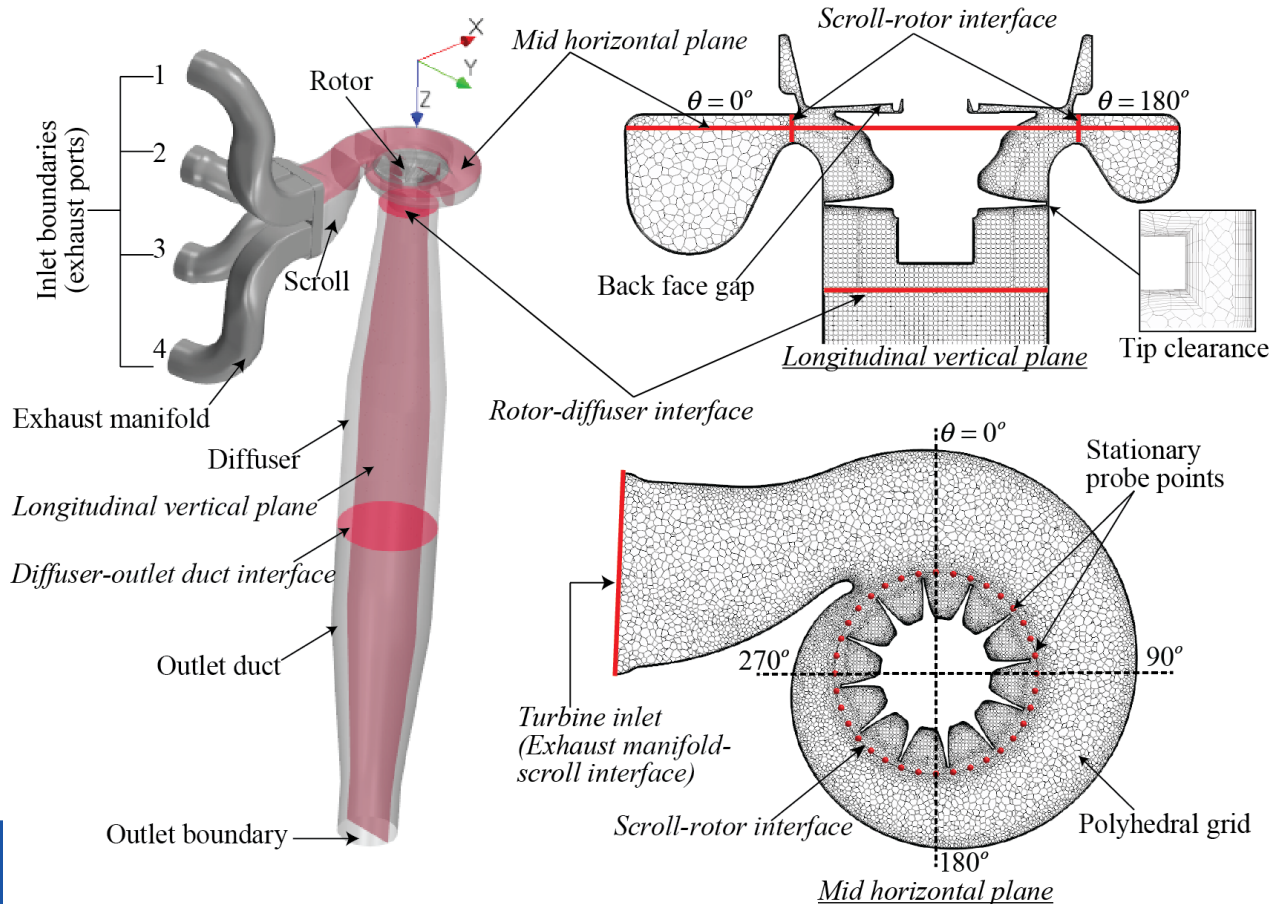
BCs



compare

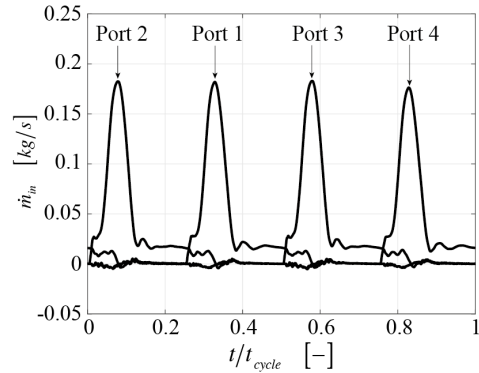
Flow fields, Performance

Computational model

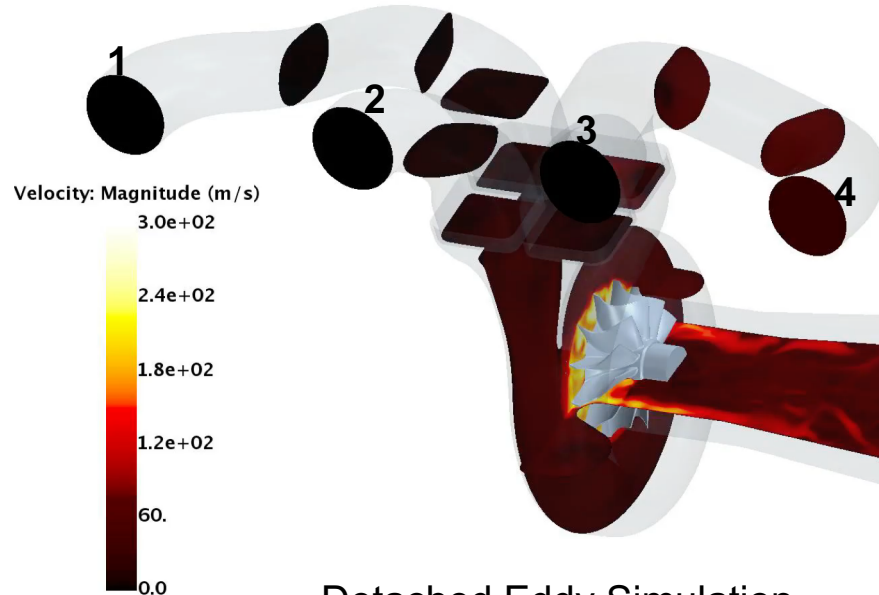
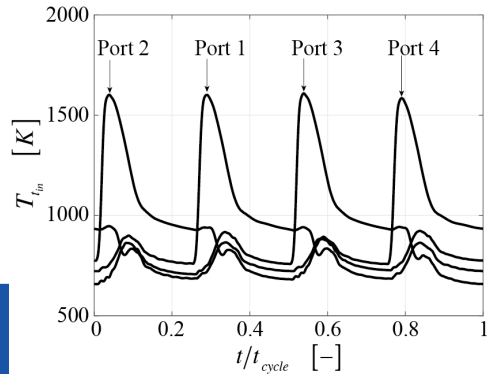


$n_{\text{engine}} = 1500 \text{ rpm}$

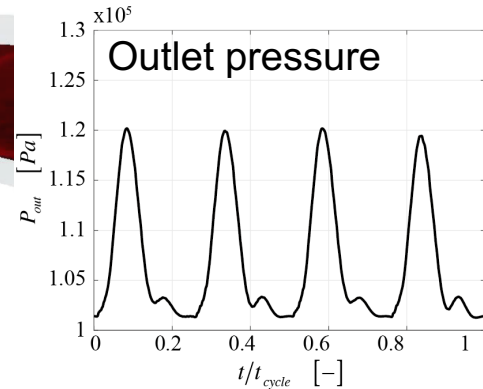
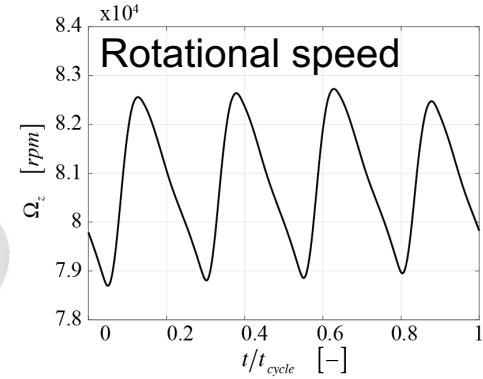
Inlet mass flow rate



Inlet total temperature



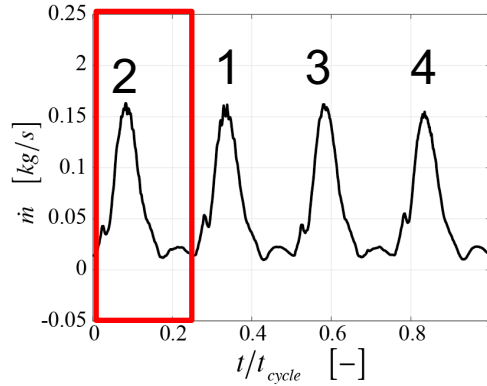
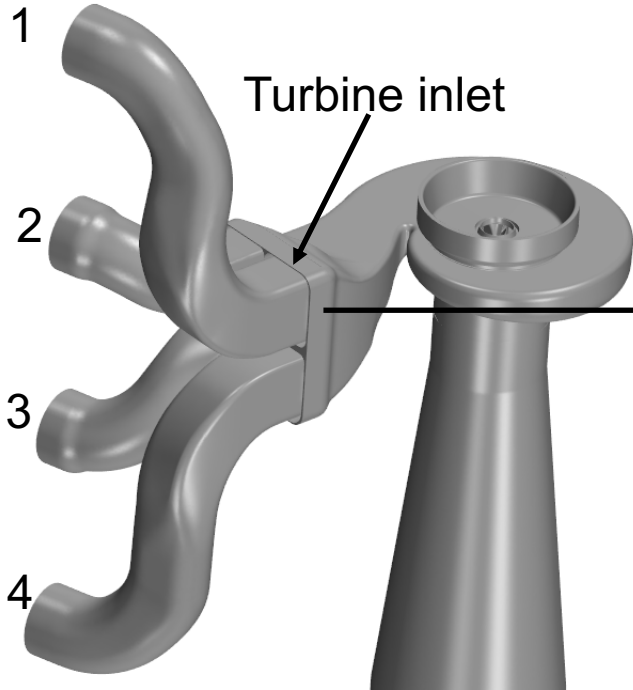
Detached Eddy Simulation (DES)



Component	Wall thermal conditions
Exhaust manifold & scroll	$T_w = 1173 \text{ K}$
Rotor	Adiabatic
Diffuser & outlet duct	$T_w = 929 \text{ K}$

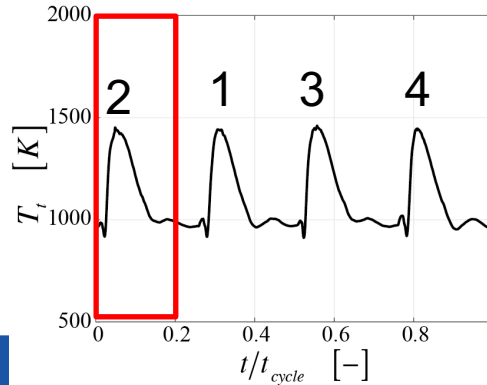
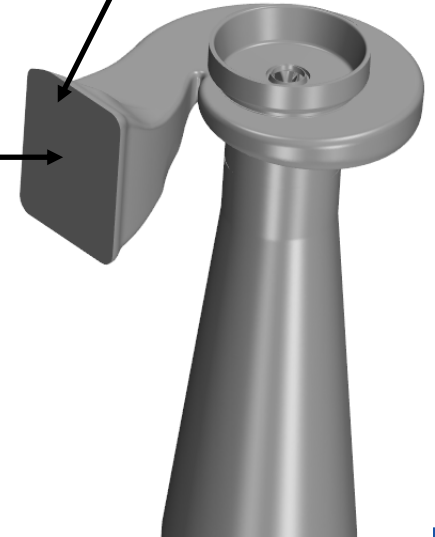
** T_w is the estimated average solid-fluid wall temperature from 1D model*

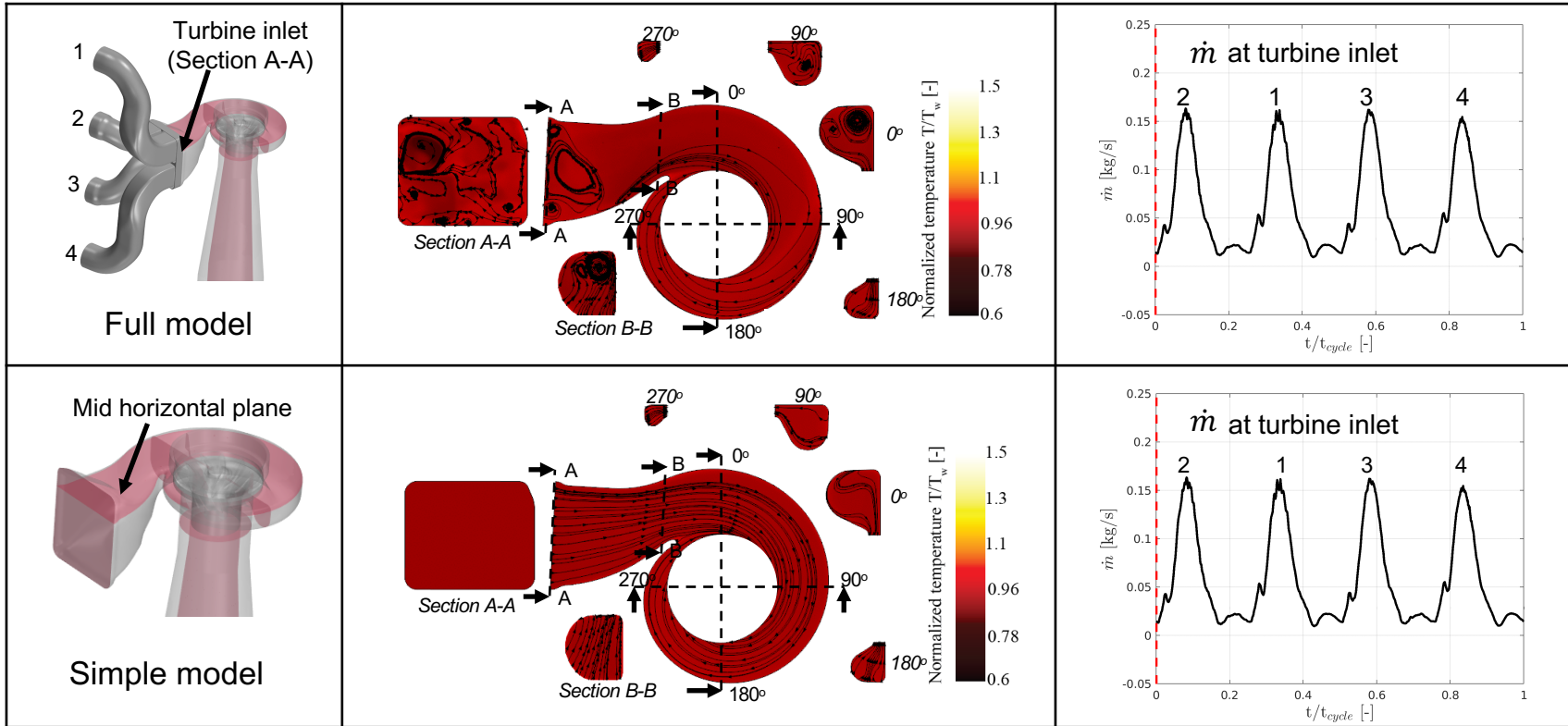
BCs: Simple model



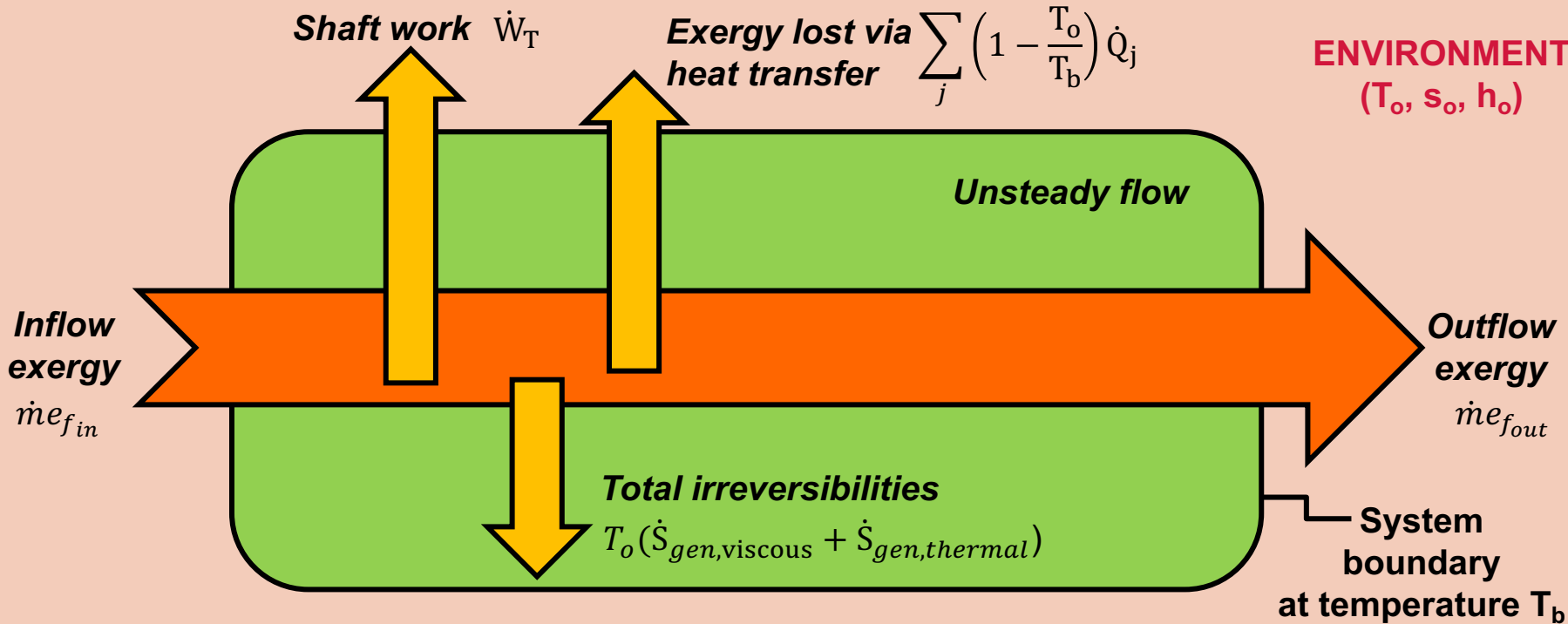
Uniform distribution

Turbine inlet





Flow exergy $e_f = [h_t(T_t) - h_o] - T_o[s(T, p) - s_o]$



$$\dot{S}_{gen,viscous} = \iiint_{V^*(t)} \left[\frac{\mu_{mol} + \mu_{turb}}{T} \Phi \right] dV$$

$$\dot{S}_{gen,thermal} = \iiint_{V^*(t)} \left[\frac{k_{mol} + k_{turb}}{T^2} (\nabla T)^2 \right] dV$$

Flow exergy budget

$$- \iint_{S^*(t)} \rho e_f (\vec{u} - \vec{u}_b) \cdot \vec{n} dS$$

$$= \frac{d}{dt} \left[\iiint_{V^*(t)} (\rho e_f) dV \right] - \frac{d}{dt} \left[\iiint_{V^*(t)} p dV \right]$$

$$+ \iint_{S^*(t)} [(\vec{r} \times \vec{f}_{net}) \cdot \vec{\Omega}] dS$$

$$+ \iint_{S^*(t)} \left[\left(1 - \frac{T_o}{T_b} \right) (\vec{q} \cdot \vec{n}) \right] dS$$

$$+ T_o (\dot{S}_{gen,viscous} + \dot{S}_{gen,thermal})$$

Physical
interpretation

Flow exergy change
(Inflow - Outflow)

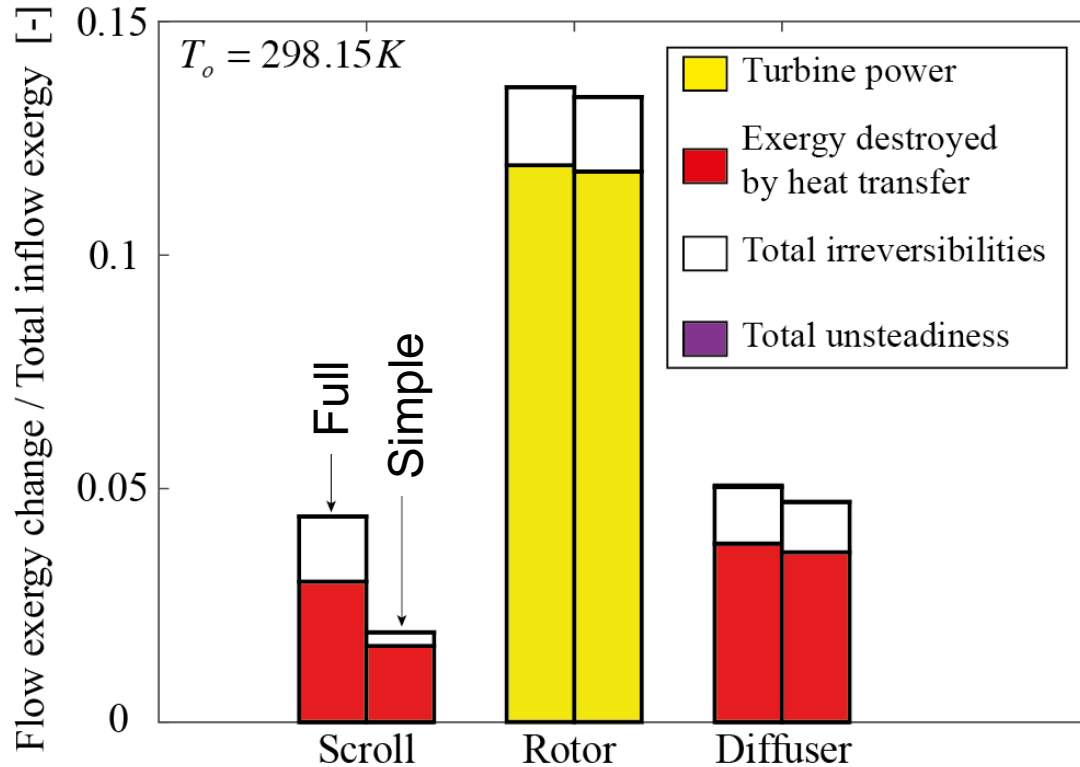
= Unsteadiness

+ Exergy transferred by shaft work

+ Exergy gained/lost by heat transfer

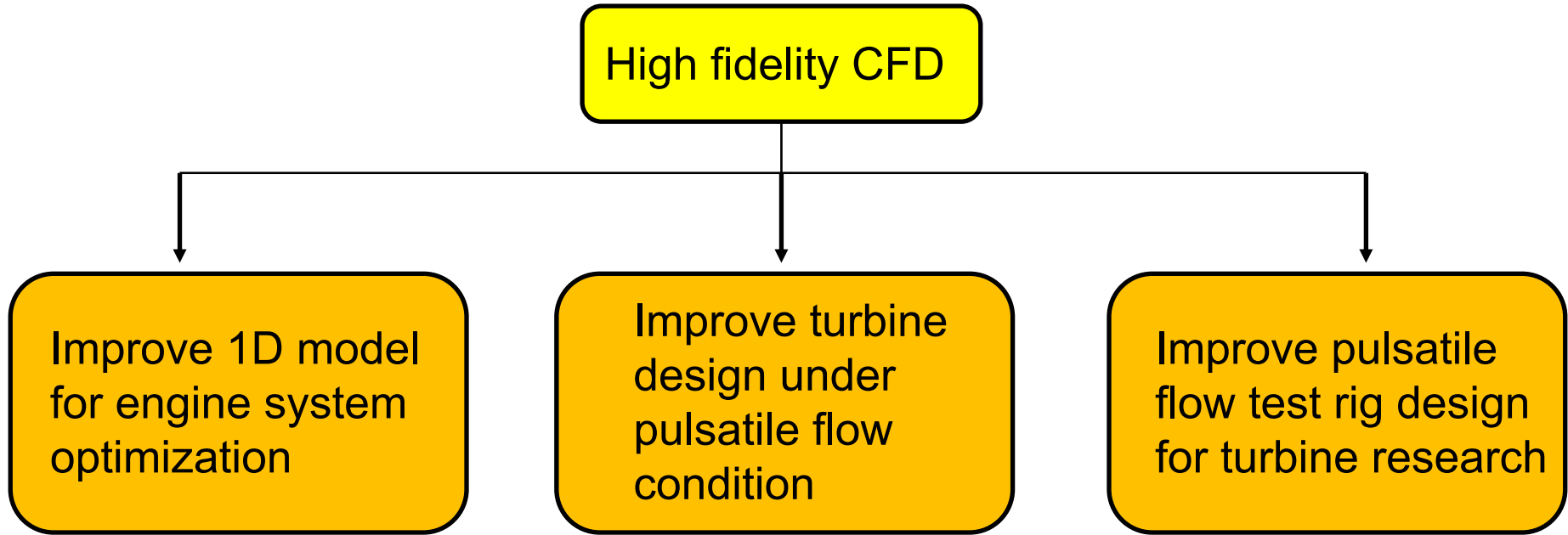
+ Exergy destroyed by irreversibilities

Flow exergy budget (1500 rpm)



- Scroll is most affected
- Turbine power is relatively unaffected

- What are the effects of upstream exhaust manifold on the heat transfer rate and performance of the turbine?
 - Fundamental thermo-fluid physics (i.e. secondary flow) in the scroll is governed by exhaust manifold
 - Heat transfer rate & aerothermodynamic losses increase
 - Turbine power is relatively unaffected





Competence Center for Gas Exchange



”Charging for the future”



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