

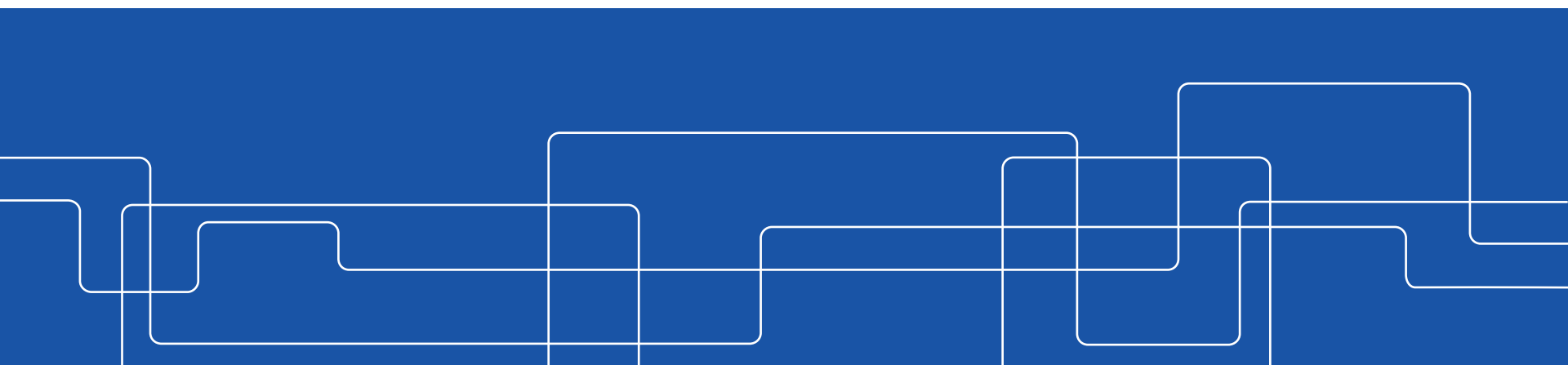


WASTE HEAT RECOVERY IN INTERNAL COMBUSTION ENGINES

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VOLVO



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- Engine's energy distribution & thermodynamic data
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- Sensitivity analysis/Parameteric effects on the Power output
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PROJECT GOAL

Design and develop a Rankine cycle (RC) system to recover unused potential heat from the Automotive ICEs with the most suitable expansion Machines

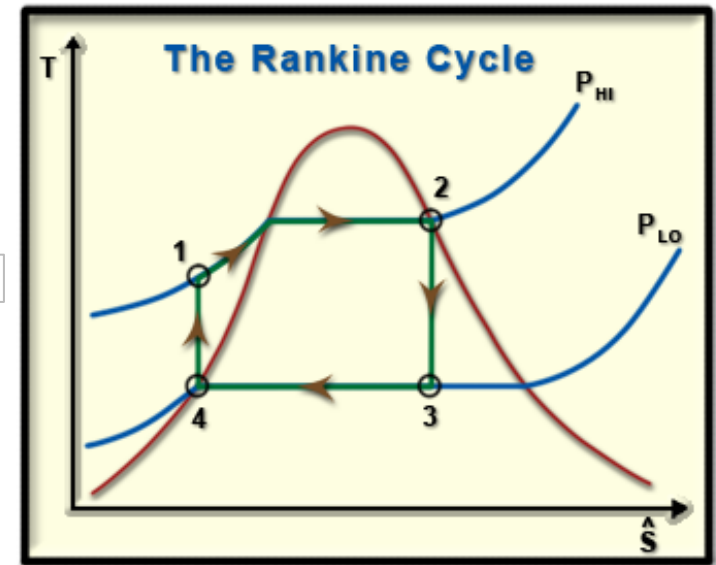
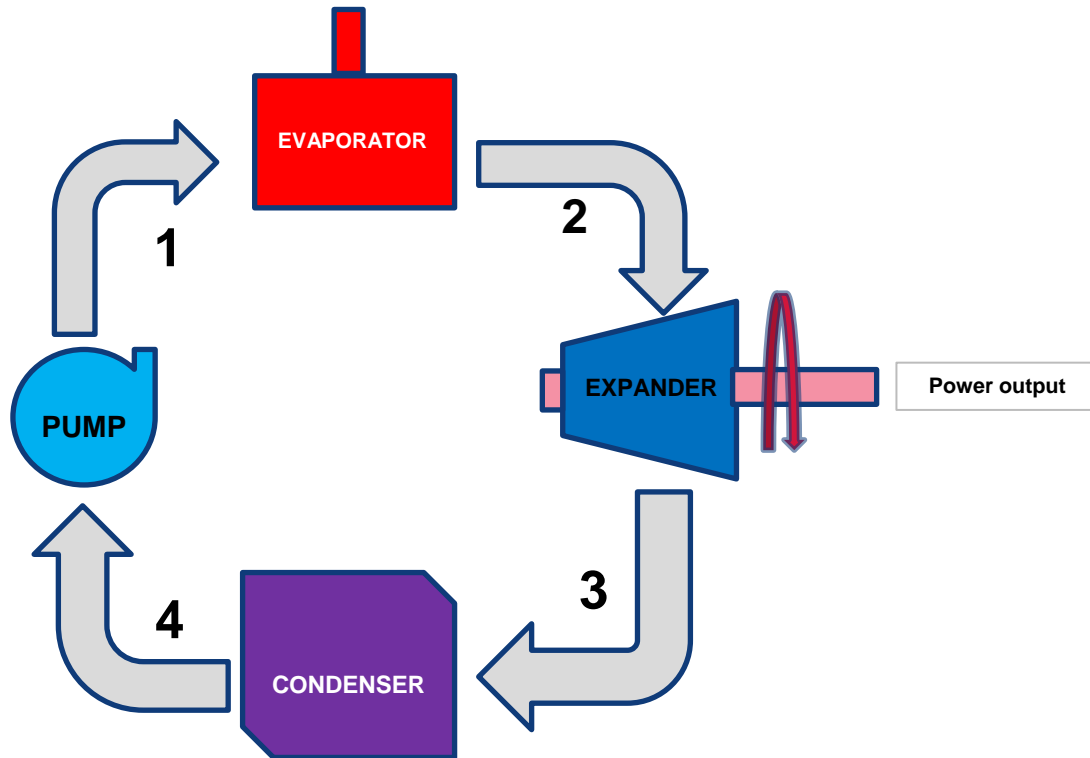
FOCUS

- Heavy duty diesel engine
- Medium and low temperature heat sources
- Expansion machines – modeling, design & development
- System integration

TODAY'S PRESENTATION

Sensitivity of the Power output towards some thermodynamic parameters

THE RANKINE CYCLE (RC) SYSTEM



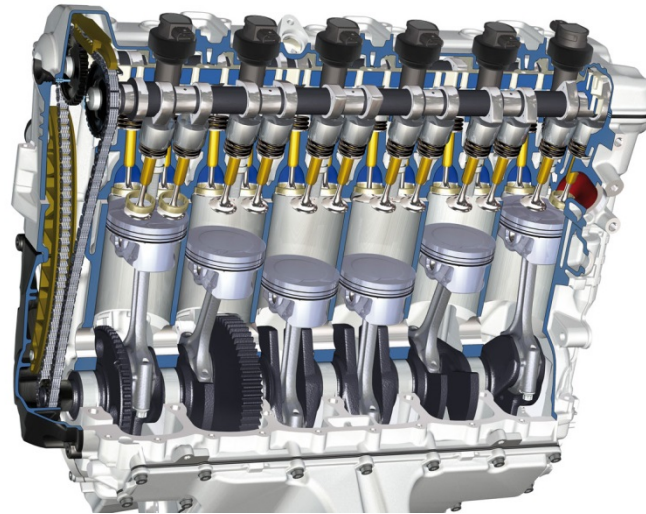
Source: <http://www.learnthermo.com>

ENGINE'S ENERGY DISTRIBUTION

Scania DL6 EU5 12,7l Heavy duty diesel engine

1100 rpm, Full load

Fuel energy: 100%



↑ Brake power:
44%

Friction & other losses: 1,42%

Coolant loss: 12%

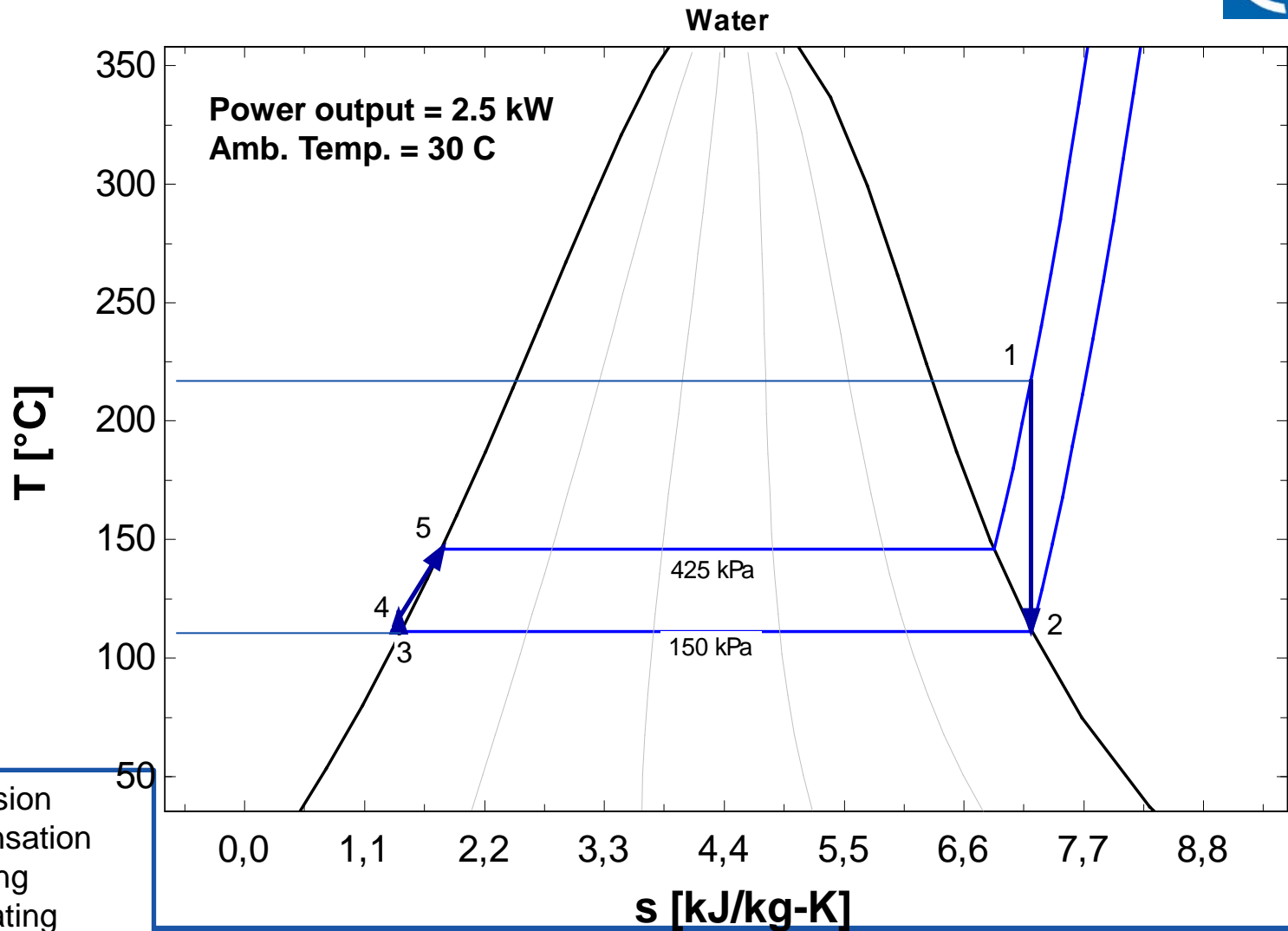
In-cylinder loss: 15%

Exhaust energy: 40%

(Picture for representation only) Source: Microsoft Powerpoint

THERMODYNAMIC PROPERTIES OF HEAT SOURCES

Heat Source	Speed	Load	Avg. T_in	Avg. T_out	Del_T	Avg. mass flow rate	Avg. Pressure	Cp,avg	Available heat
	RPM	%	C	C	C	g/s	bar	kJ/kg.K	kW
Exhaust	1100	35	330,53	30	301	144	1,02	1,091	47,21
EGR	1100	35	343	102	341	45	1,58	1,102	11,95
CAC	1100	35	76	25,57	50,42	139	1,57	1,007	7



- | | |
|-----|--------------|
| 1-2 | Expansion |
| 2-3 | Condensation |
| 3-4 | Pumping |
| 4-5 | Preheating |
| 5-1 | Vaporisation |

PRELIMINARY RC OPERATING CONDITIONS (EXHAUST)

<u>PARAMETER</u>	<u>UNIT</u>	<u>VALUE</u>
Q_absorbed	kW	46,18
Pressure ratio		2.8
Expander isentropic & mech. efficiency		0,8 (assumed)
Pump isentropic efficiency		0,8 (assumed)
Mass flow rate_ of_working fluid	g/s	18,99 (actual)
Expander inlet volume flow rate	m ³ /h	35,85
Specific volume at turbine inlet	m ³ /kg	0,52
Expansion ratio		2,2
Thermal efficiency of the cycle	%	5,39

PRELIMINARY SENSITIVITY ANALYSIS

@ FIXED AVAILABLE HEAT (WATER)

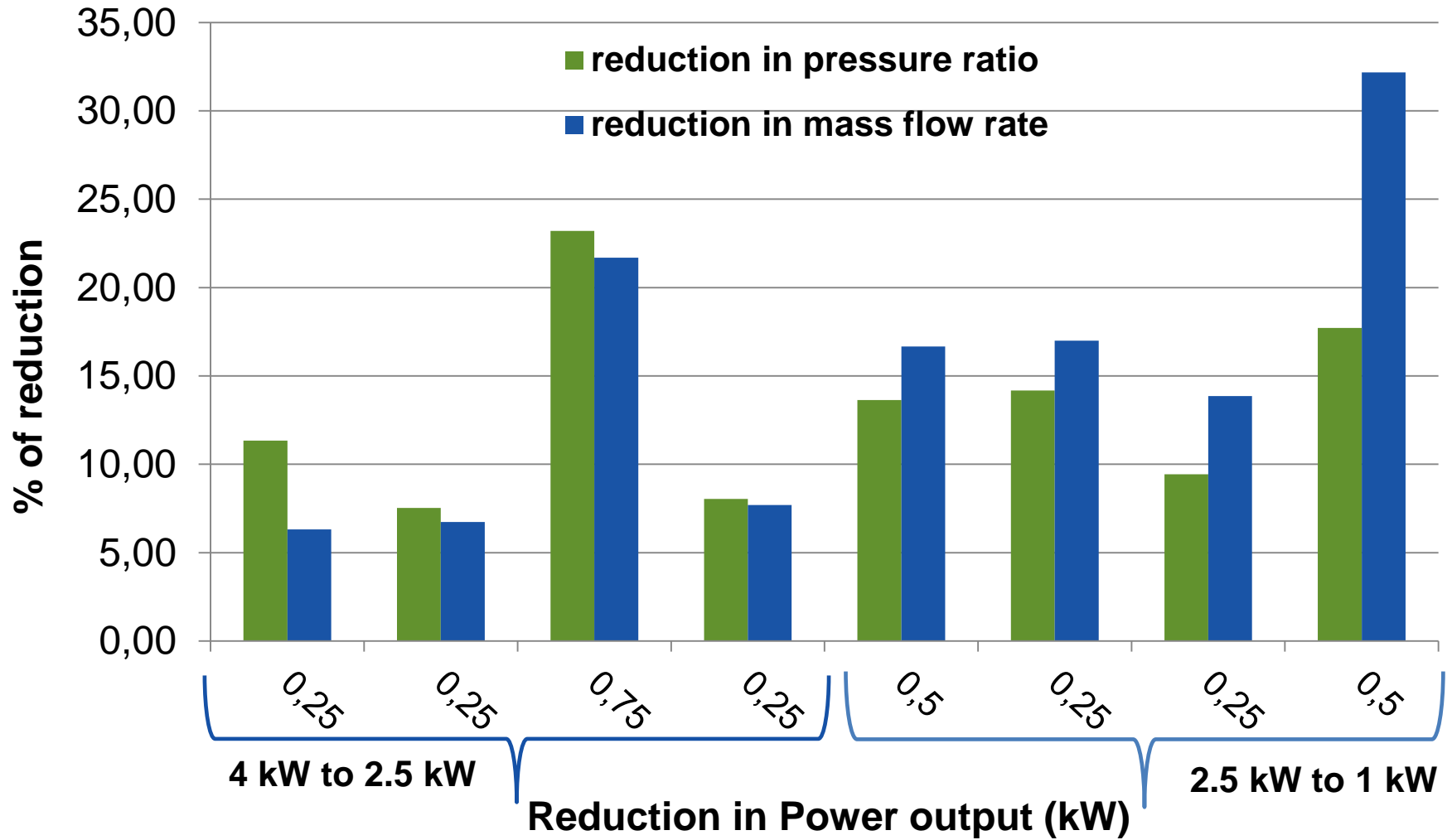
Mass flow rate (CONSTANT)	Pressure ratio (PR)	Reduction in PR	Enthalpy drop ↓	Reduction in enthalpy drop	Thermal efficiency ↓	Power output ↓
g/s	↓	%	kJ/kg	%	%	kW
19,5	4,94		329,34		8,13	4
19,5	4,38	11,34	300,35	8,80	7,5	3,75
19,5	4,05	7,53	281,35	14,57	7,08	3,5
19,5	3,7	8,64	260,49	20,91	6,6	3,25
19,5	3,11	15,95	220,96	32,91	5,69	2,75
19,5	2,86	8,04	202,77	38,43	5,26	2,5
19,5	2,47	13,64	170,99	48,08	4,49	2
19,5	2,12	14,17	139,83	57,54	3,71	1,75
19,5	1,92	9,43	120,23	63,49	3,22	1,5
19,5	1,58	17,71	81,9	75,13	2,22	1

PRELIMINARY SENSITIVITY ANALYSIS

@ FIXED AVAILABLE HEAT (WATER)

Mass flow rate m_wf ↓	Reduction in m_wf	Pressure ratio (PR) (CONSTANT)	Enthalpy drop (CONSTANT)	Thermal efficiency (CONSTANT)	Heat absorbed, Q_in ↓	Reduction in Q_in	Power output ↓
g/s	%		kJ/kg	%	kW	%	kW
19		4,94	329,34	8,13	49,1		4
17,8	6,32	4,94	329,34	8,13	46	11,03	3,75
16,6	6,74	4,94	329,34	8,13	42,9	17,02	3,5
15,25	8,13	4,94	329,34	8,13	36,8	28,82	3,25
13	14,75	4,94	329,34	8,13	33,6	35,01	2,75
12	7,69	4,94	329,34	8,13	31	40,04	2,5
10	16,67	4,94	329,34	8,13	25,9	49,90	2
8,3	17	4,94	329,34	8,13	21,5	58,41	1,75
7,15	13,86	4,94	329,34	8,13	18,5	64,22	1,5
4,85	32,17	4,94	329,34	8,13	12,5	75,82	1

PARAMETRIC EFFECTS ON THE POWER OUTPUT



PARAMETER EFFECTS – SUMMARY

- ★ Work outputs higher than 2 & 2,5 kW
 - ★ *Mass flow rate & Enthalpy drop dominate (resp.)*
- ★ Lower work outputs
 - ★ *Pressure ratio dominates*

↑ High mass flow rate ↑ Higher heat input ↑ More heat recovery

At constant pressure ratio,

↑ High mass flow rate
↑ higher heat input
↑ higher work output
↔ same efficiency

CONCLUSION

- Parametric/Sensitivity analysis may help in the choice of expanders
- Drawbacks to be addressed in the sensitivity analysis e.g. pinch-point
- Detailed analysis of the parametric effects on '*Expansion*' required
- Comparison of performance of other fluids with water



Competence Center for Gas Exchange



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