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Control of particle agglomeration with relevance to after-treatment gas processes

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Outline



Motivation of the project
Description of the model
Results
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Future plans



Motivation





NOMAD

(https://commons.wikimedia.org/wiki/File:Traffic jamdelhi.jpg), "Trafficjamdelhi", https://creativecommons.org/licenses/by/2.0/le galcode

> Ruben de Rijcke (https://commons.wikimedia.org/wiki/File:Automobile_exhaust_gas.jp g), "Automobile exhaust gas", https://creativecommons.org/licenses/by-sa/3.0/legalcode

- Nano sized particles in emissions from internal combustion engines(ICE) are a major health issue.
- Larger particles are easier to filtrate.
- Particle agglomeration is one way in which larger particles can be obtained from ICE particulate emissions.





Scope of the study



- (Why?)Reduce the number of particles in the internal combustion engine(ICE) exhaust gases.
- (How?)Using flow and acoustic forcing to enhance particle agglomeration.
- (Insight)Perform numerical studies to study particle behavior under pulsatile flow conditions. Make comparison against measurements on an actual engine exhaust system.
- (Goal)Utilize the insight to help the industry develop a suitable prototype that can be used as an after treatment device.



Model assumptions and agglomeration principle



- 1D in nature.
- Laminar and incompressible flow field.
- Stokesian regime is assumed. The equation however can be extended to non-Stokesian regime by considering drag coefficients.

In this **1D** model we have:

- Oscillations with respect to time.
- Oscillations with respect to geometry.

These oscillations will cause particles to accelerate or decelerate based on their location.

Pipe and corresponding 1D schematic λ velocity Inlet Outlet $t = t_0$ time Initial 1d setup $t = t_0 + n_q \Delta t \quad \mid n_q \in \mathbb{N}$ Grouping $t = t_0 + m_g \Delta t \quad \begin{array}{c} m_g \in \mathbb{N} \\ m_g > n_g \end{array}$ Grouping _____ $t = t_0 + n_{nq} \Delta t \quad n_{nq} \in \mathbb{N}$ Non-Grouping $t = t_0 + m_{ng} \Delta t \quad \begin{bmatrix} m_{ng} \in \mathbb{N} \\ m_{ng} > n_{ng} \end{bmatrix}$ Non-Grouping

Pipe length is considered to be 1m.

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Picture of grouping vs. non grouping

Grouping: all particle trajectories collapse into one of the two bands and particle velocities stay the same.



Non grouping: particle trajectories move individually and particle velocities differ.





Description of model equation

• The model gas velocity equation is given as:

$$\begin{split} v_g(x,t) &= \underbrace{V_a}_{a} - V_b \left(\underbrace{\sin(\omega t + \phi)}_{\text{Mean}} \right) \underbrace{\cos(kx)}_{\text{Mean}} \\ \text{Mean} \\ \text{Part.} \\ engine. \\ \end{split} \\ \end{split}$$

• The (non-dim)equation for the motion of particles in such a flow field:

$$\ddot{x}^* = \frac{1}{\underline{St}} \left(v_g^* - \dot{x}^* \right)$$

Stokes number controls inertia.

□ Enforce the restrictions $V_a \ge V_b$ and $V_a + V_b \le 80m/s$.



Important Parameters



- Two parameters were identified by Katoshevski et.al(2005) as being important.
- The beta parameter given as:

$$\beta = \frac{V_a - \lambda f}{V_b} = \frac{V_a^* - 1}{V_b^*}. \quad \left\{ \begin{array}{l} \text{Mean flow vs.} \\ \text{Oscillations.} \end{array} \right\}$$

• The alpha parameter given as:

$$\alpha = \frac{1}{\sqrt{(V_b^*St)}} \cdot \left\{ \begin{array}{l} \text{Size of the} \\ \text{particles.} \end{array} \right\}$$

Likely indicators of grouping $|\beta| \le 1$ and α should be large.



Parameter Studies





Trend shows that the highest grouping points for high wavelengths(0.1 and 0.12m).

□ It is important to think about the design in order to be in the right operating points.



Pulse synchronization







Grouping case with $V_a = 21$ m/s, $V_b = 19$ m/s, $\lambda = 0.1$ m and R.P.M value of 5200.

Non grouping case with $V_a = 26$ m/s, $V_b = 10$ m/s, $\lambda = 0.1$ m and R.P.M value of 5200.

□In the case of grouping pulses are synchronized.



Summary



- Conclusion of the 1D study and submission of manuscript for SAE World Congress.
- Verification/Validation study using a BFS case by Fessler et. al 99.



Future plans



- Introduce particles inside **3D** geometries.
- Perform high fidelity studies in 3D using OpenFOAM.





"Charging for the future"









BorgWarner

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Illustration of equilibrium points' scenario



Model gas velocity: $v_g(x,t) = V_a - V_b (\sin(kx - \omega t))$

Non-dim gas velocity: $v_g^*(x,t) = V_a^* - V_b^* (\sin(x^* - t^*))$

Model gas velocity equal to V_w in non dim form:

$$1 = V_a^* - V_b^* (\sin(x^* - t^*))$$

□Normalization of time, displacement and velocity is done using ω , *k* and $\omega/_k = V_w$.



Grouping criterion used



- Particle velocities are observed at different points.
- If the following criteria is satisfied at all the aforementioned observed points then that case is considered as grouping:

$$\sqrt{\left(v_{p1} - v_{p2}\right)^2 + \left(v_{p2} - v_{p3}\right)^2 + \dots + \left(v_{p8} - v_{p9}\right)^2} < 1$$

where v_{pi} denotes the particle velocity of the *i*'th particle.



Backward facing step validation study



	^	
$ \begin{array}{c} \text{Inlet} \\ & \underbrace{10H} \\ & \underbrace{H} \end{array} \xrightarrow{1} \end{array} $	2.5H	Outlet

RANS k-omega SST model.
BC: 1) Inlet: Velocity Dirichlet(9.37,0,0)m/s, Pressure zero Gradient.
2) Outlet: Velocity zeroGradient, Pressure Dirichlet(zero).
3) Walls: No Slip Dirichlet(zero). Periodic in the spanwise direction
No. of Cells: ~3mil



Comparison of CFD with experiments





Mean streamwise velocity profiles downstream of the step are shown. U_0 is the centerline velocity.



Parameter Studies

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